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(12) **United States Patent**
Cheyne et al.

(10) **Patent No.: US 9,222,243 B2**
(45) **Date of Patent: Dec. 29, 2015**

(54) **WEAR ASSEMBLY**

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(73) Assignee: **ESCO Corporation**, Portland, OR (US)

(*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 505 days.

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Related U.S. Application Data

(60) Provisional application No. 61/507,726, filed on Jul. 14, 2011, provisional application No. 61/576,929, filed on Dec. 16, 2011.

(51) **Int. Cl.**
E02F 9/28 (2006.01)

(52) **U.S. Cl.**
CPC ***E02F 9/2883*** (2013.01); ***E02F 9/2825*** (2013.01); ***E02F 9/2833*** (2013.01); ***E02F 9/2841*** (2013.01); ***E02F 9/2858*** (2013.01)

(58) **Field of Classification Search**
CPC E02F 9/28; E02F 9/2825; E02F 9/2833; E02F 9/2841; E02F 9/2858; E02F 9/2883
USPC 37/452, 453, 455, 456, 906
See application file for complete search history.

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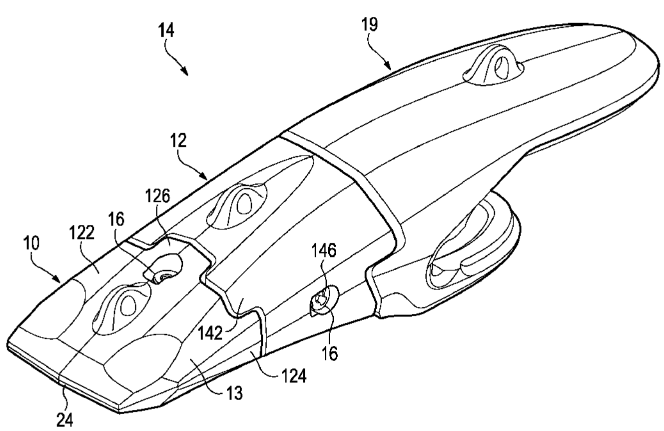
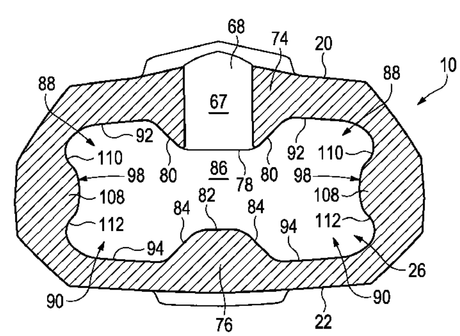
Primary Examiner — Jamie L McGowan
(74) *Attorney, Agent, or Firm* — Steven P. Schad; John Anderton

(57) **ABSTRACT**

A wear assembly for use on various kinds of earth working equipment that includes a base with a supporting portion, a wear member with a cavity into which the supporting portion is received, and a lock to releasably secure the wear member to the base. The supporting portion is formed with top and bottom recesses that receive complementary projections of the wear member. These recesses and projections include aligned holes so as to receive and position the lock centrally within the wear assembly and remote from the wear surface. The lock includes a mounting component that defines a threaded opening for receiving a threaded pin that is used to releasably hold the wear member to the base. The separate mounting component can be easily manufactured and secured within the wear member for less expense and higher quality than forming the threads directly in the wear member.

5 Claims, 21 Drawing Sheets





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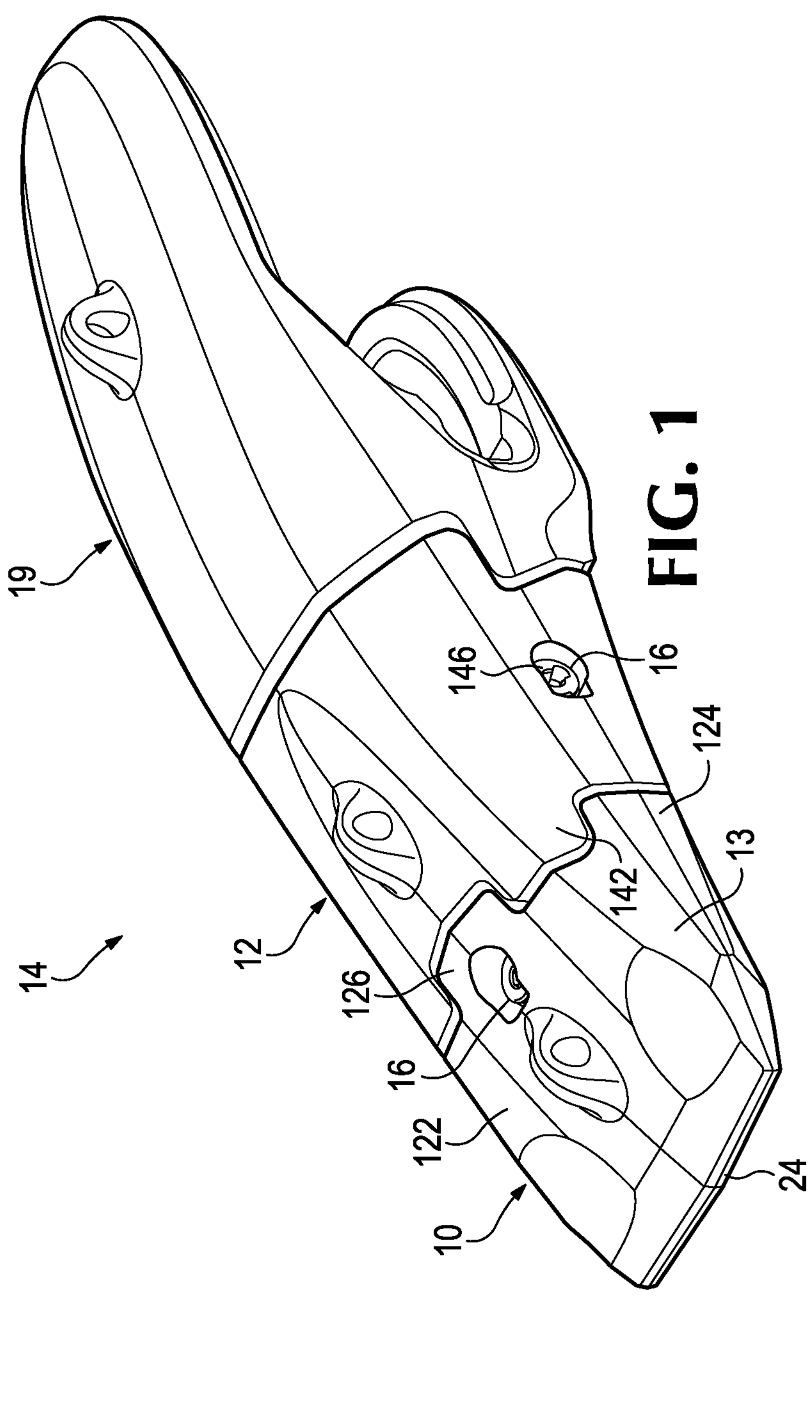

FIG. 1

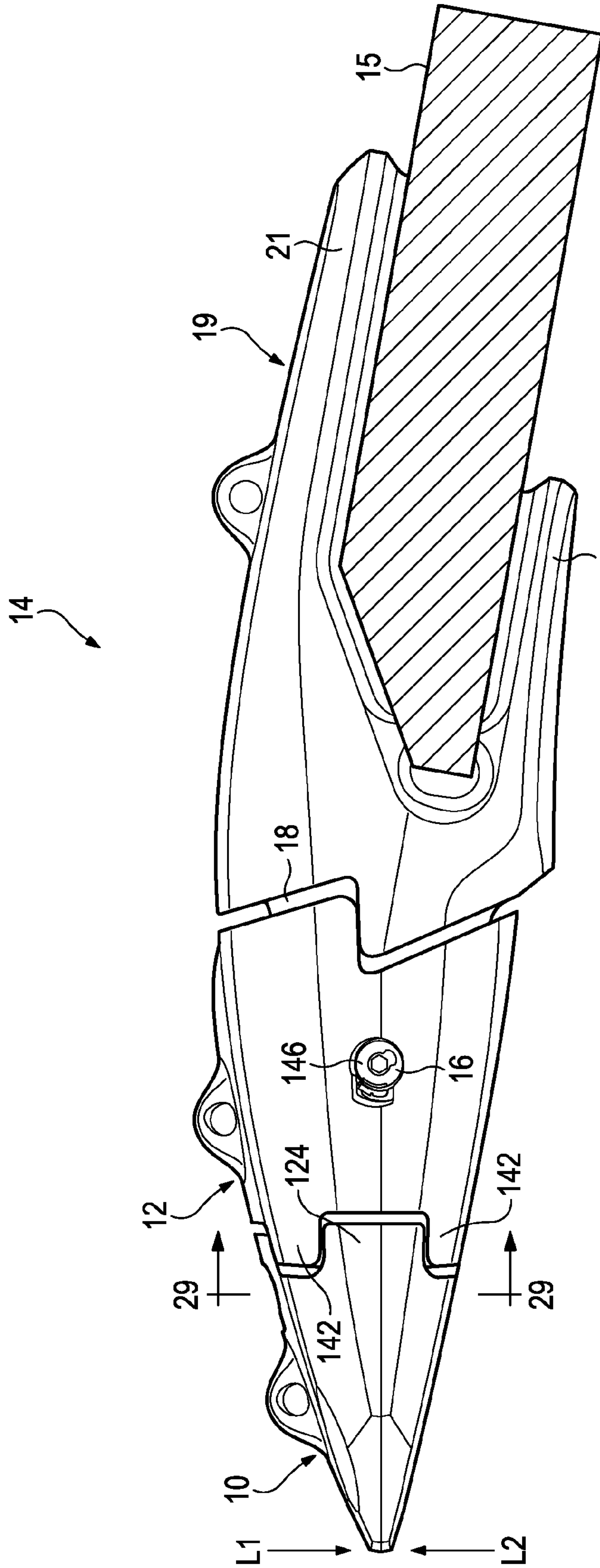

FIG. 2

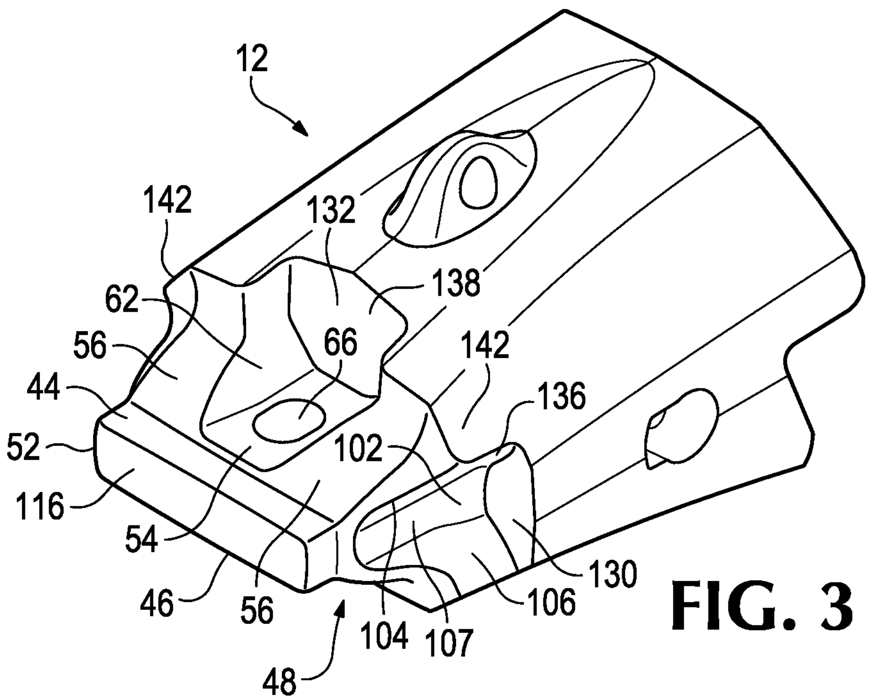

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FIG. 3

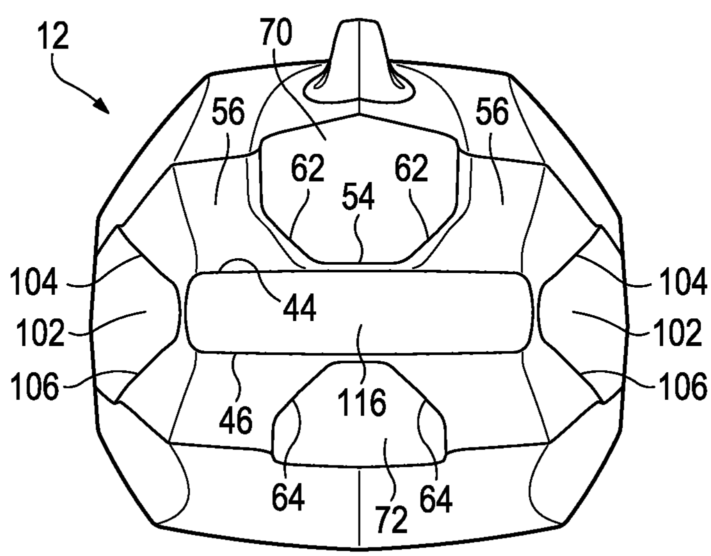

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FIG. 4

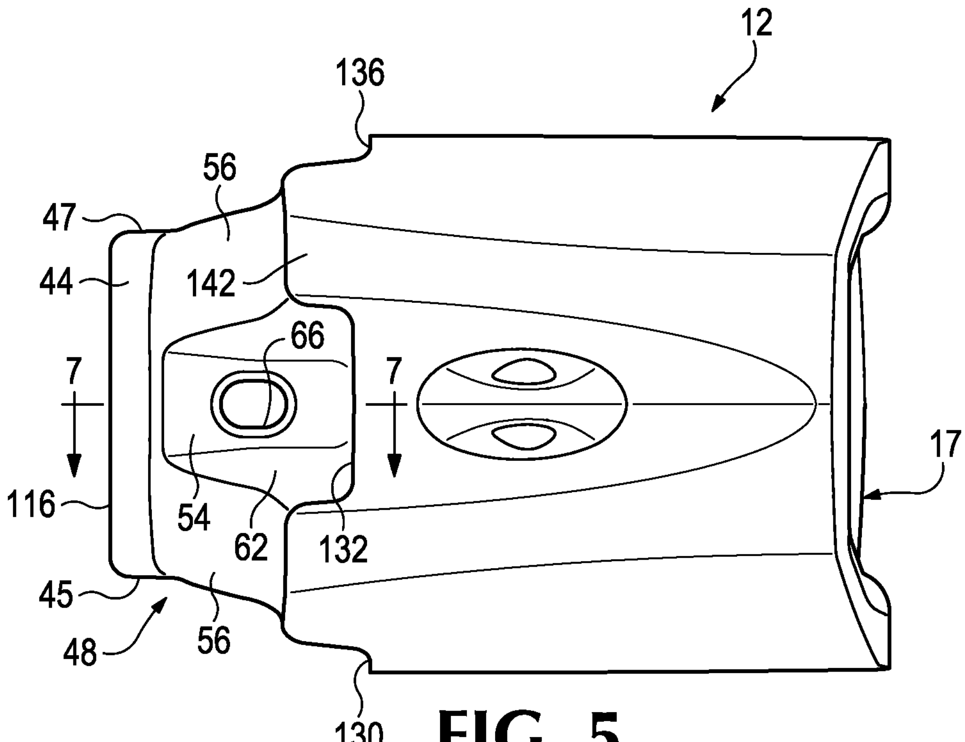
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FIG. 5

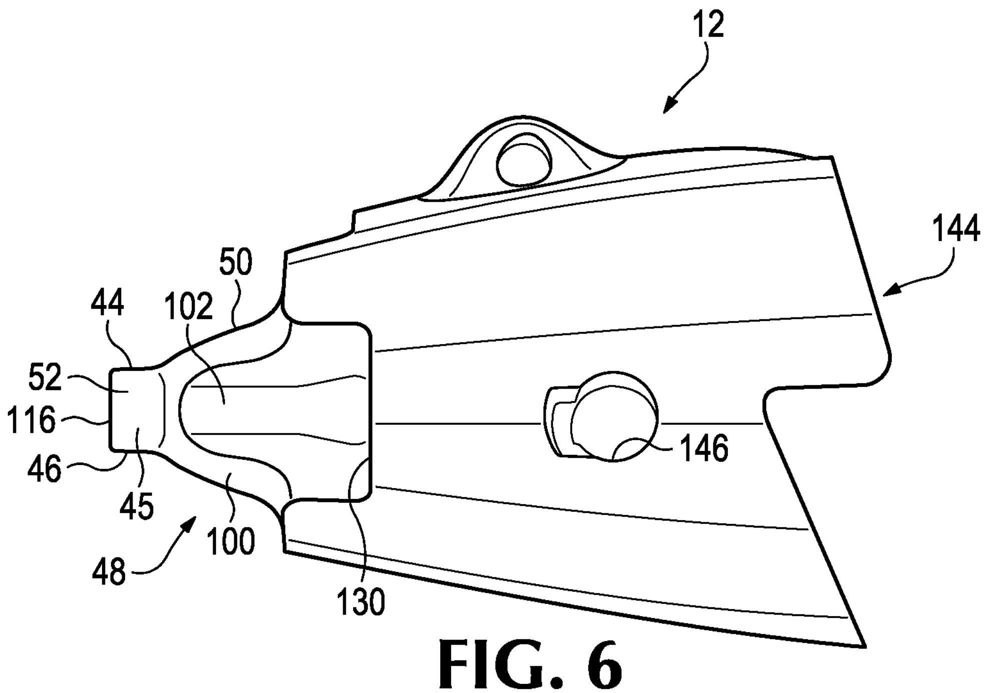
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FIG. 6

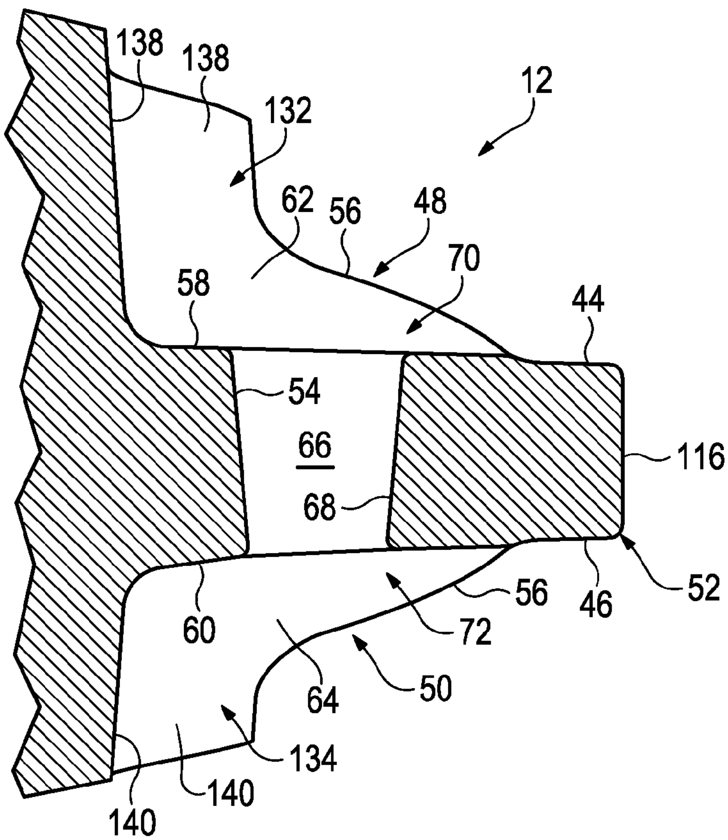
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FIG. 7

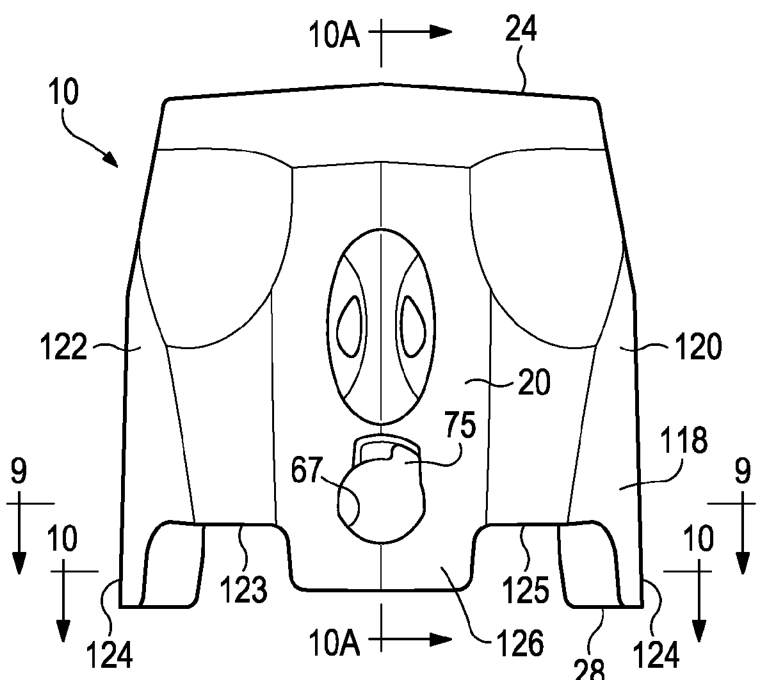
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FIG. 8

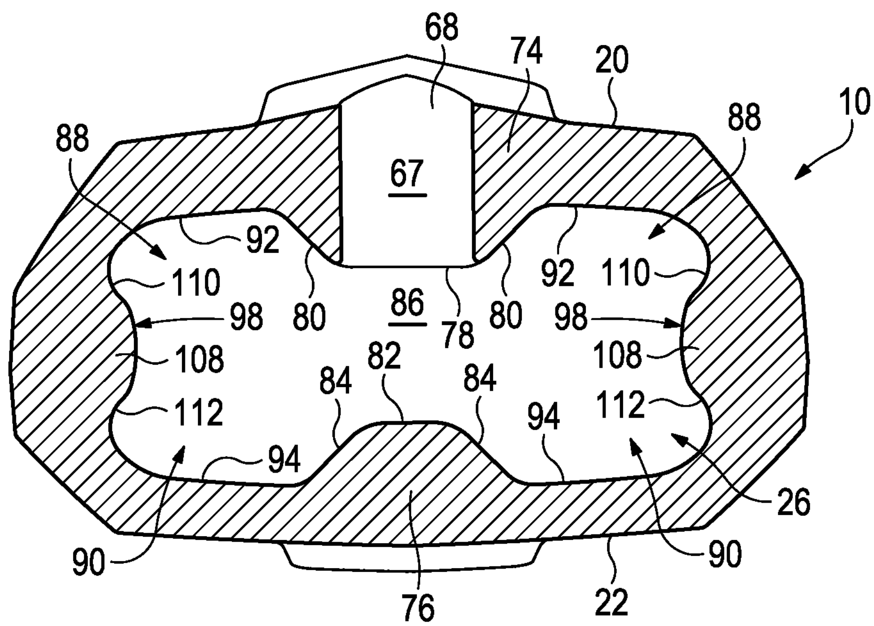
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FIG. 9

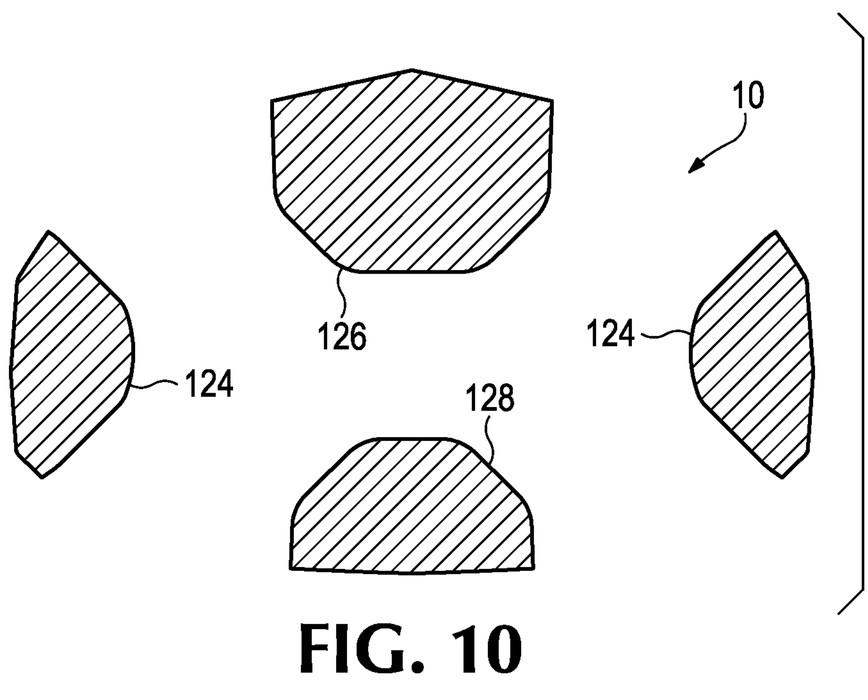
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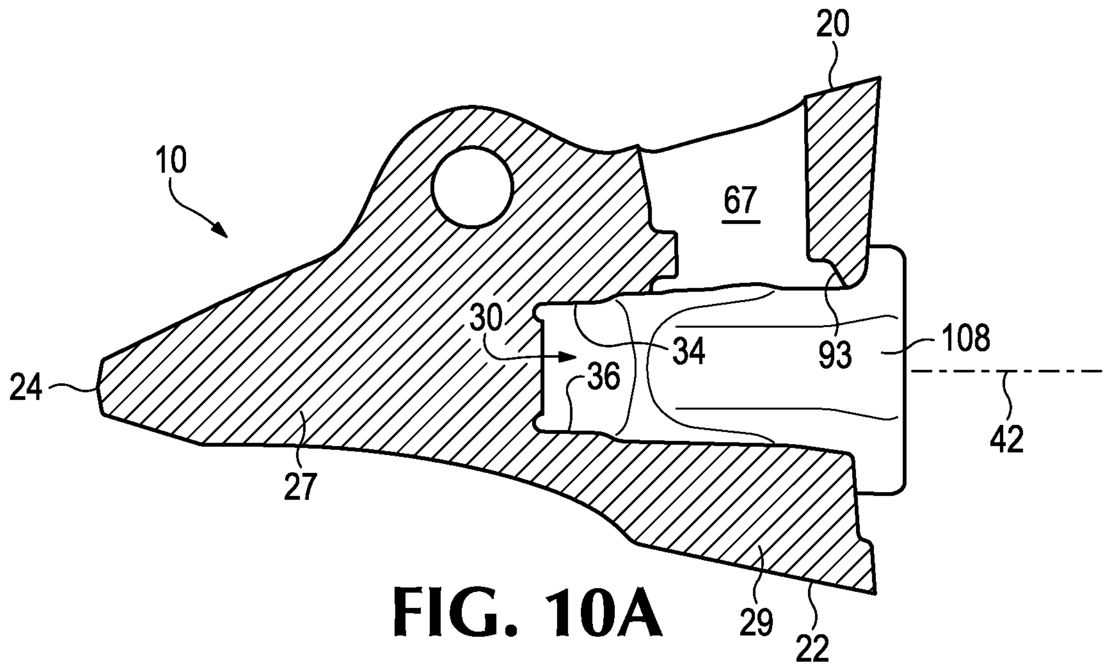
FIG. 10



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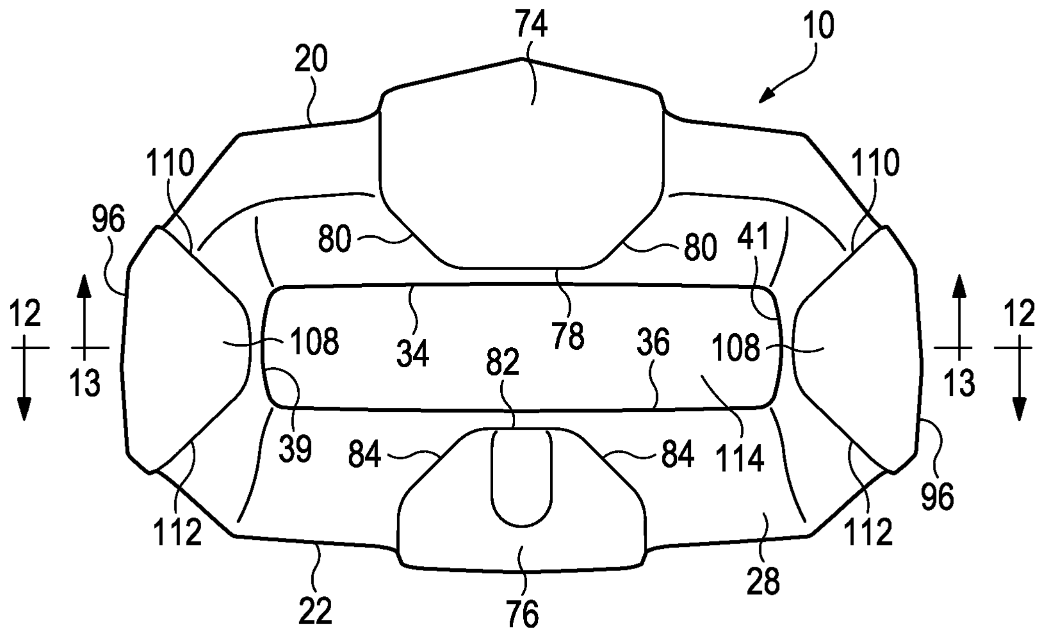

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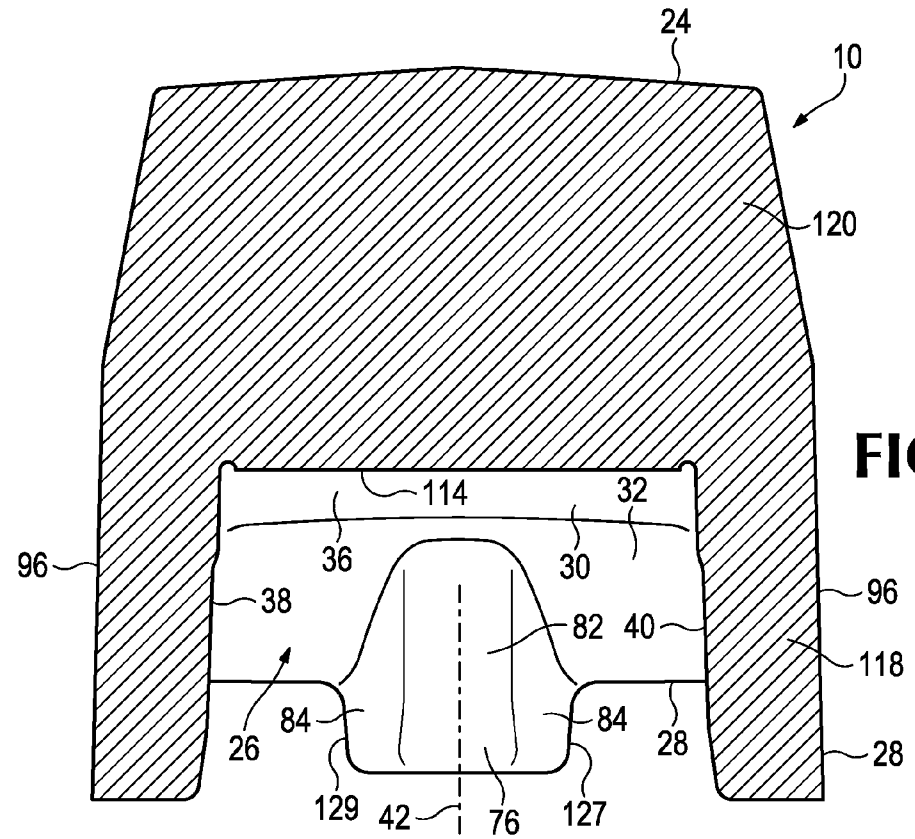
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FIG. 12

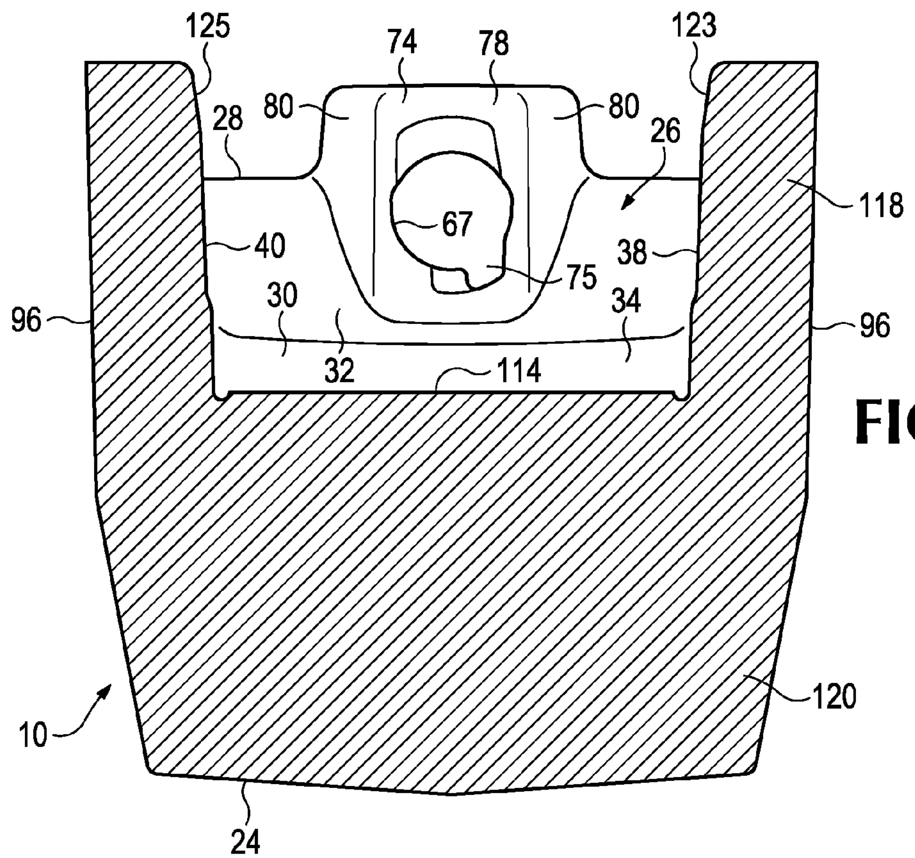
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FIG. 13

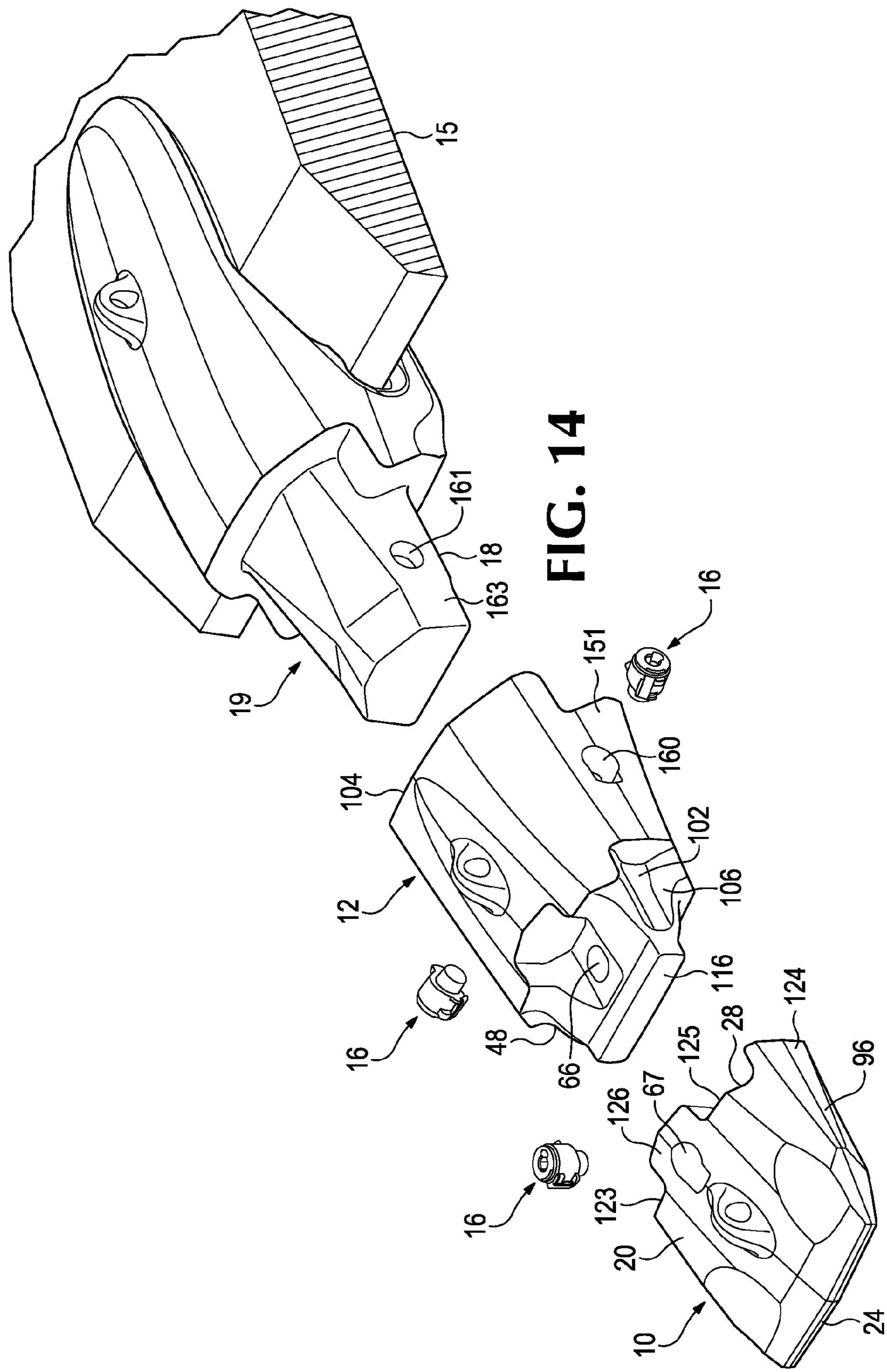
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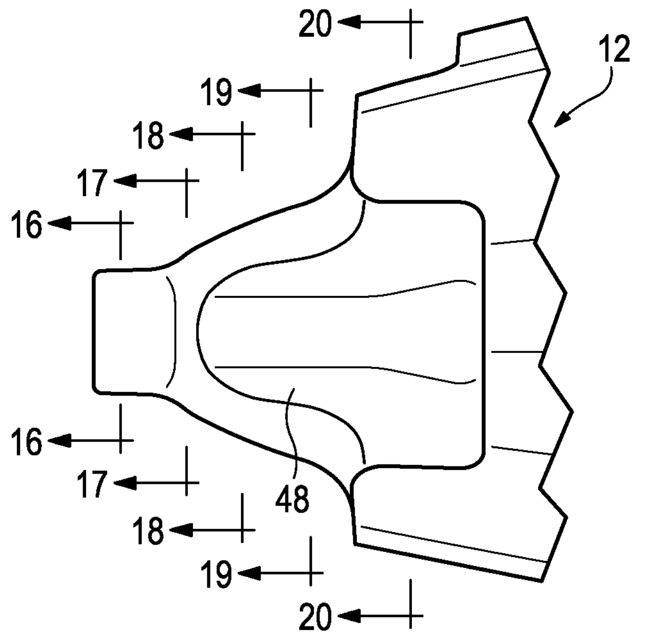
FIG. 14



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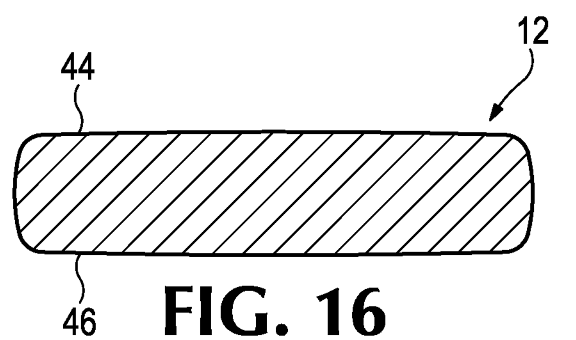

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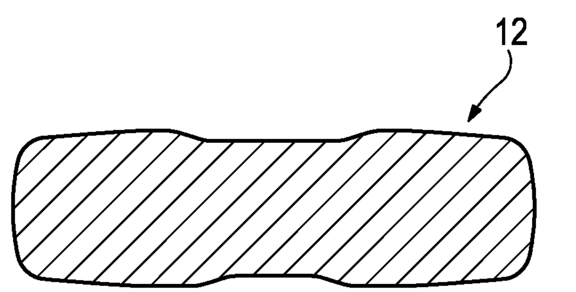

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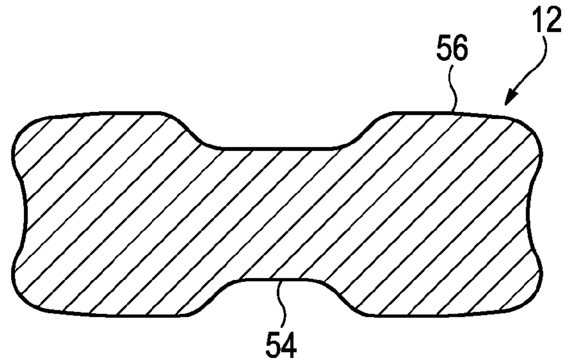

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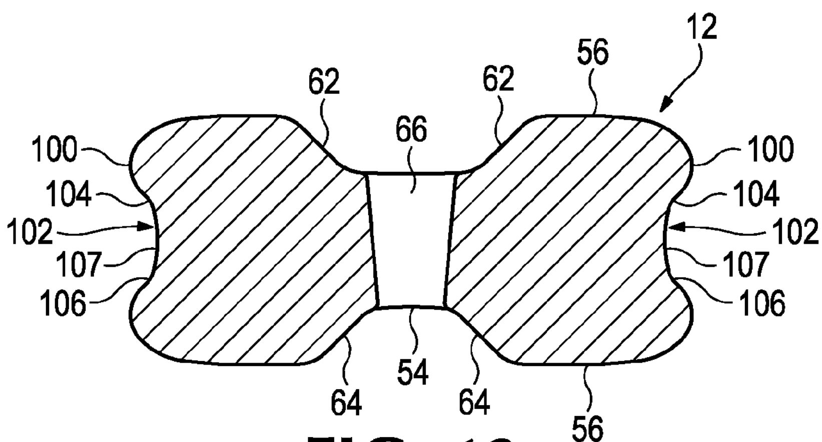

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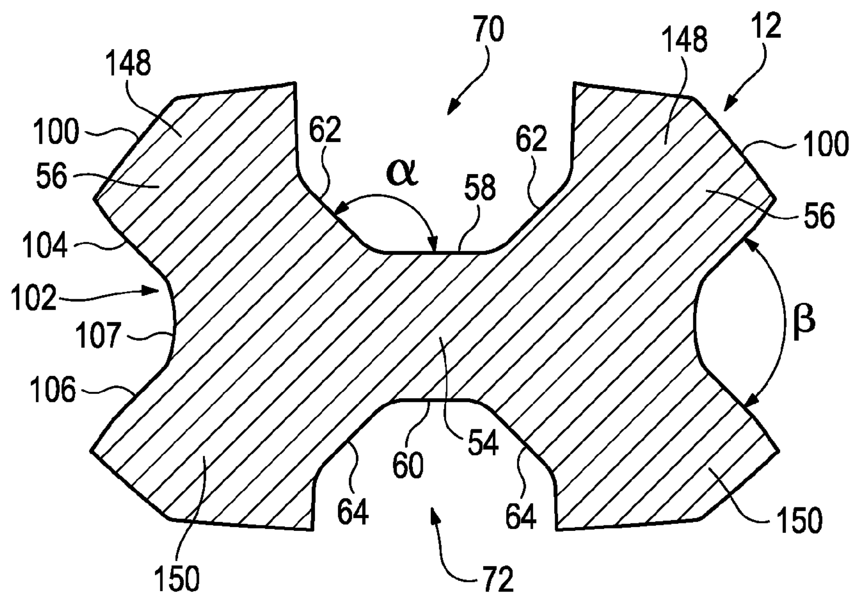

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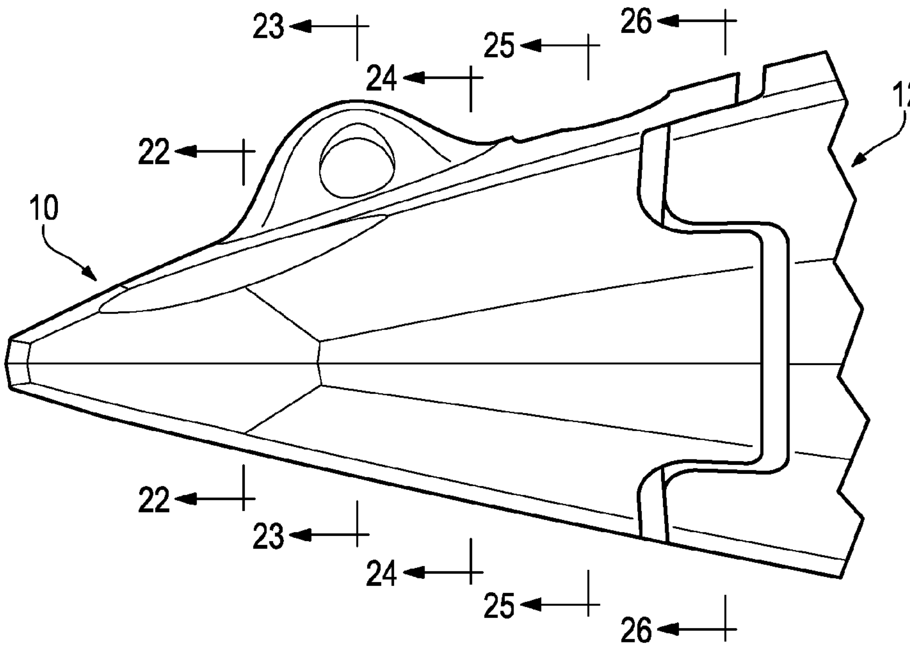

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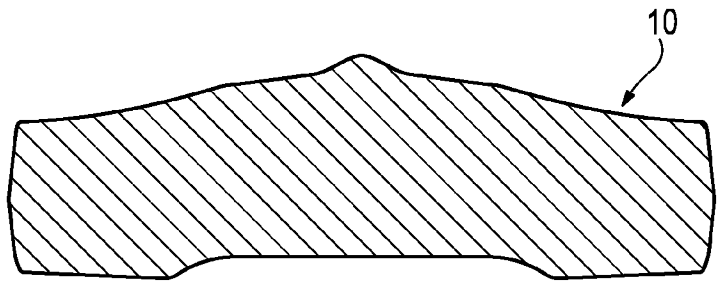
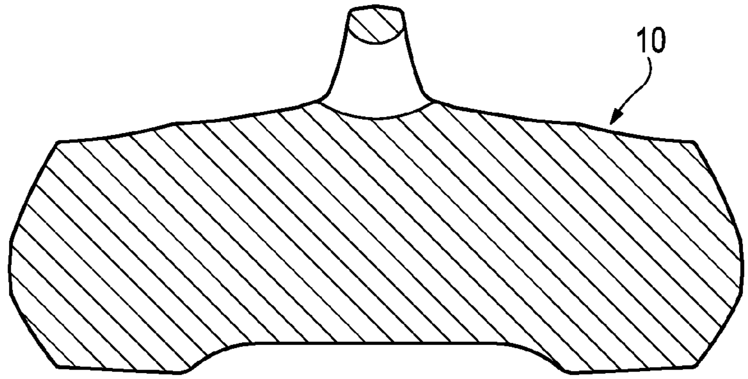


FIG. 22



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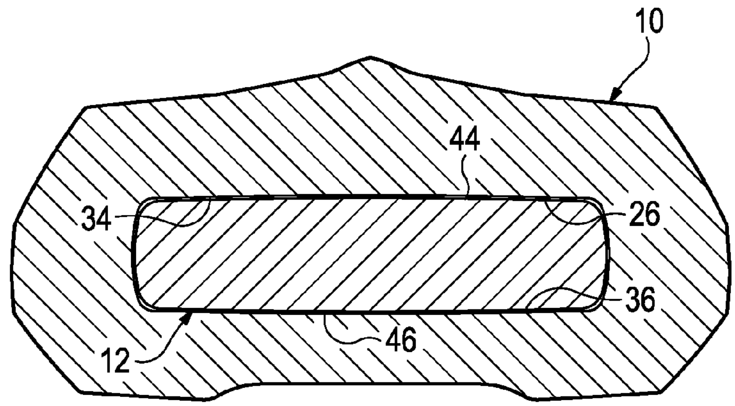

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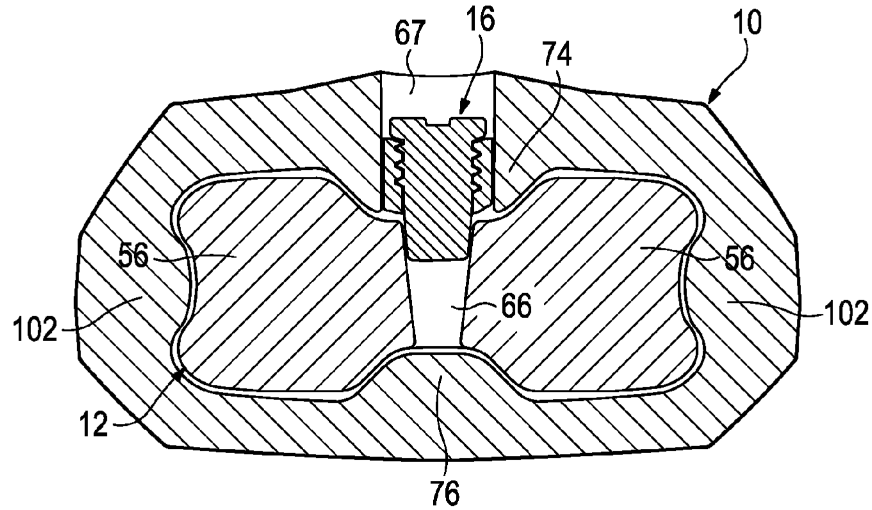

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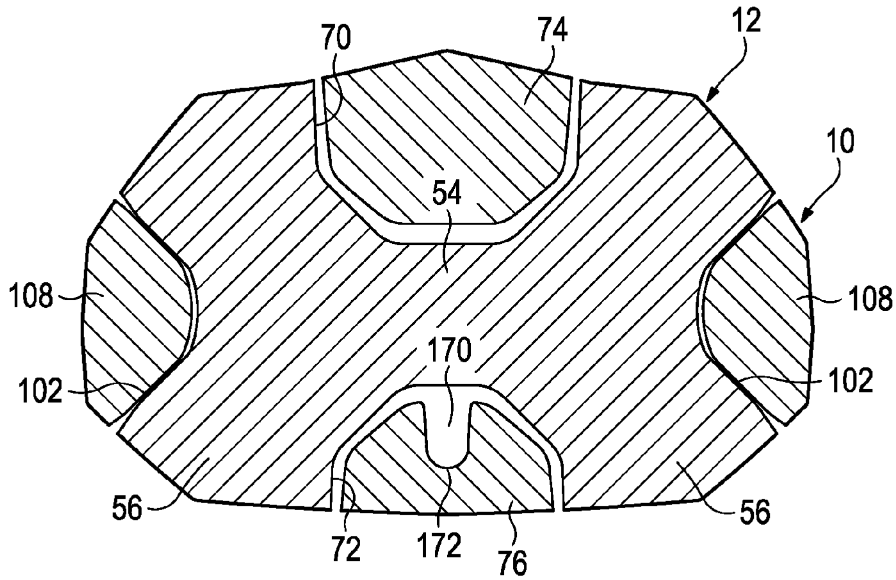

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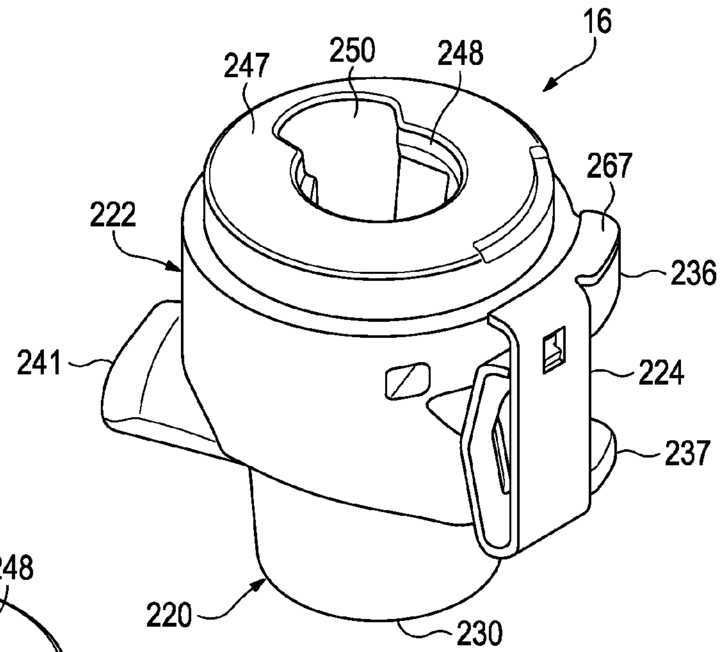
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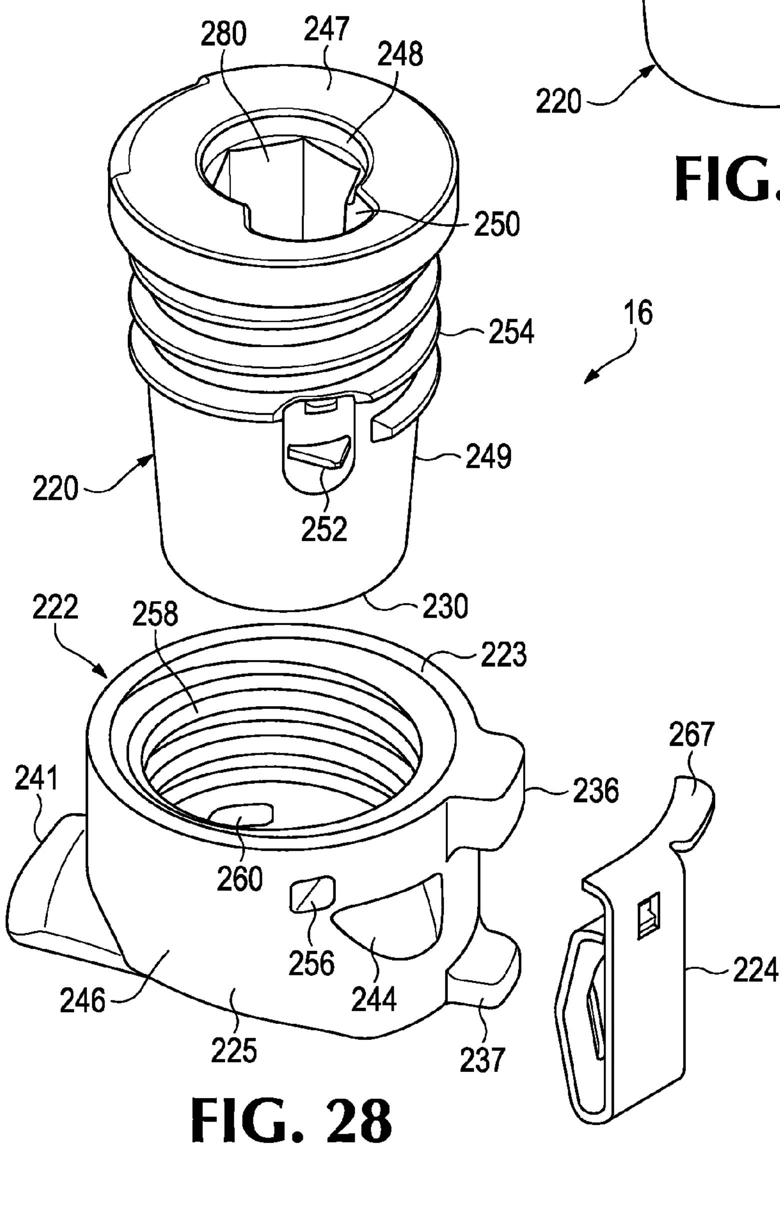
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FIG. 28

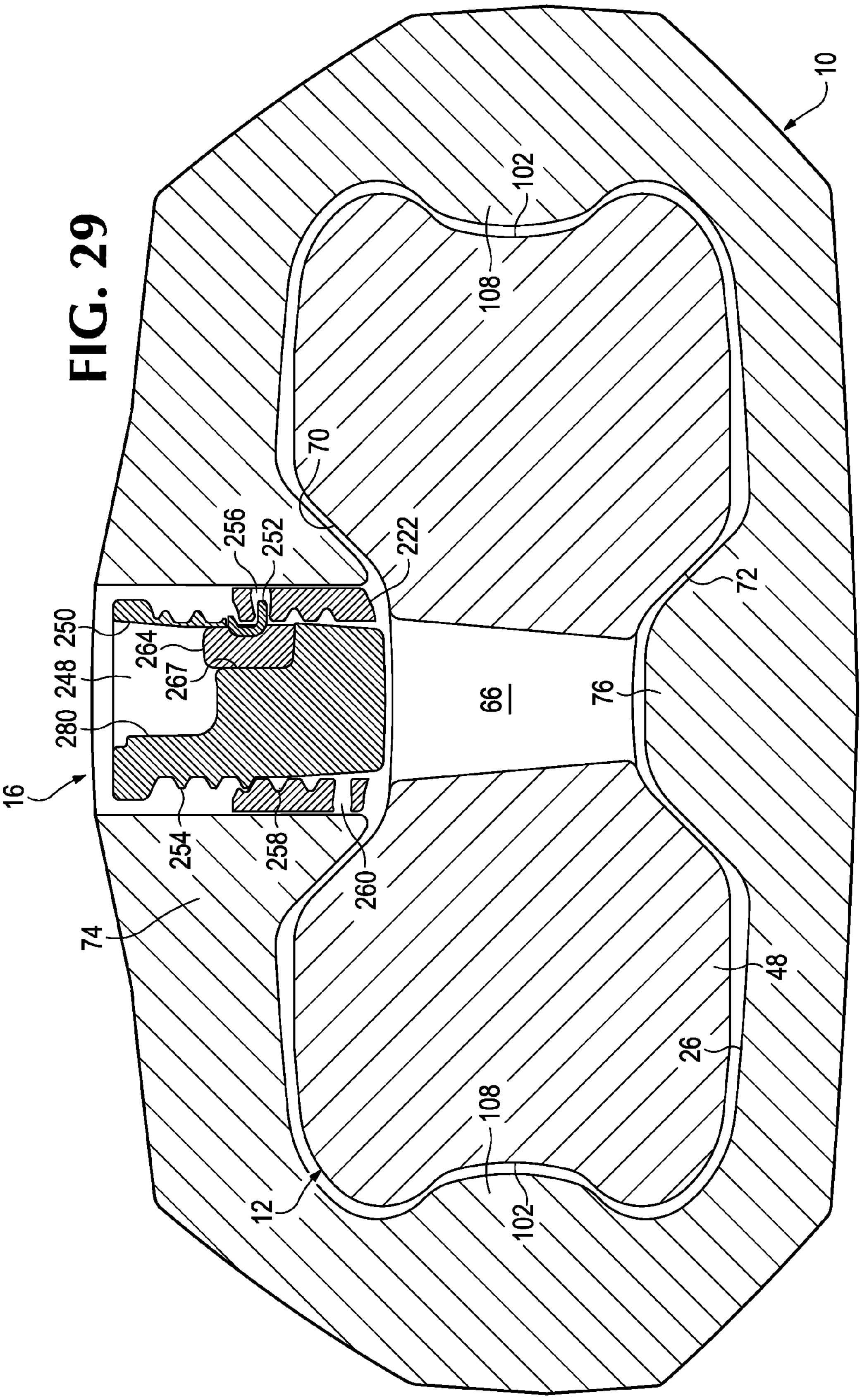
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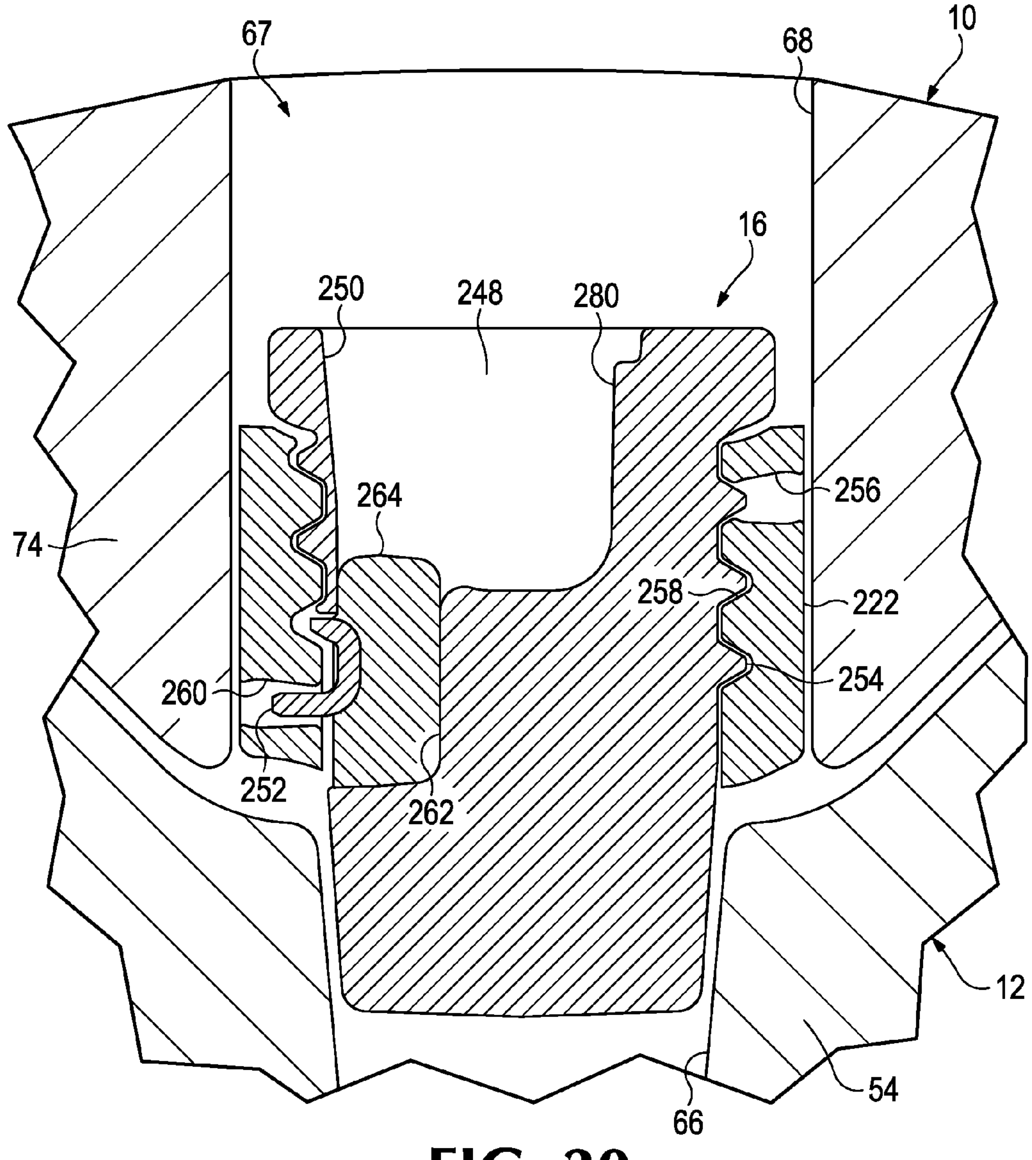
FIG. 29



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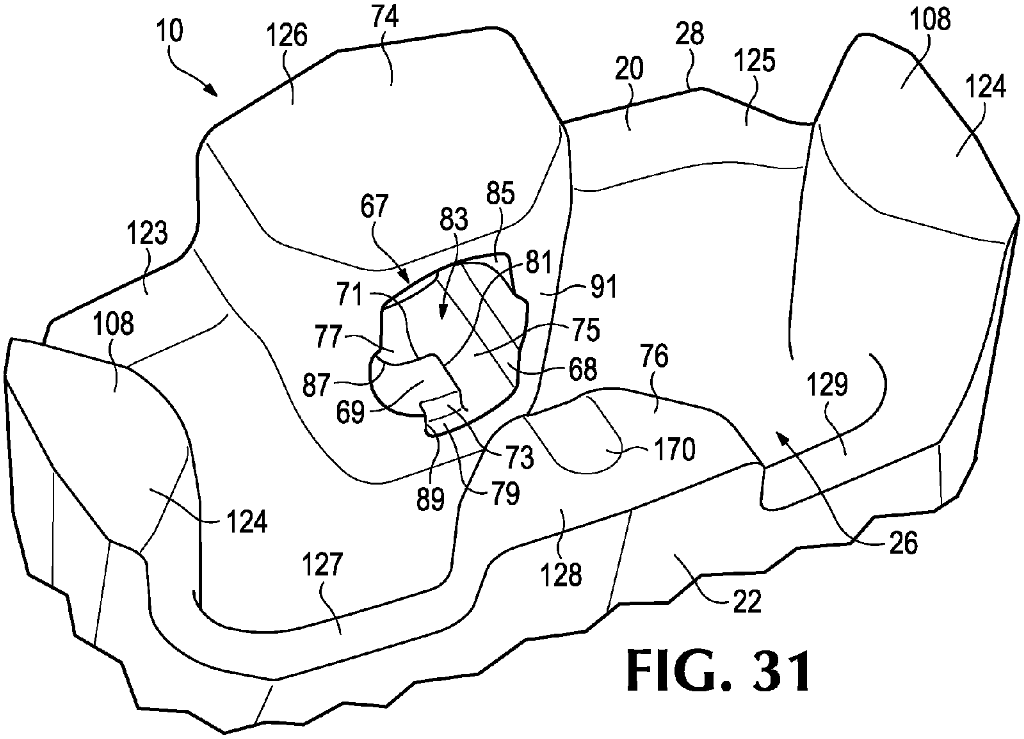

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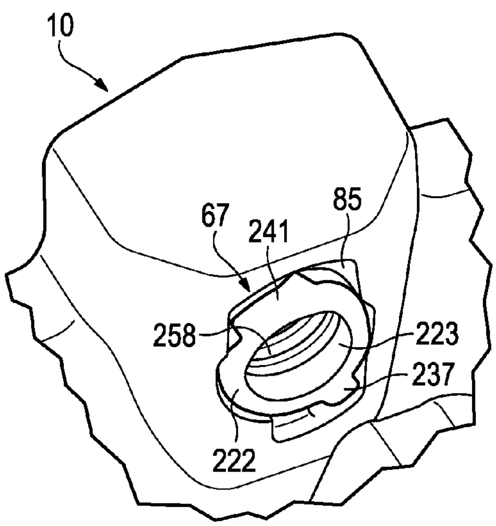

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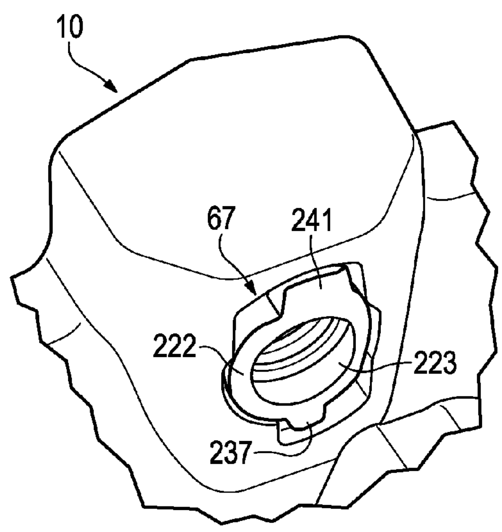

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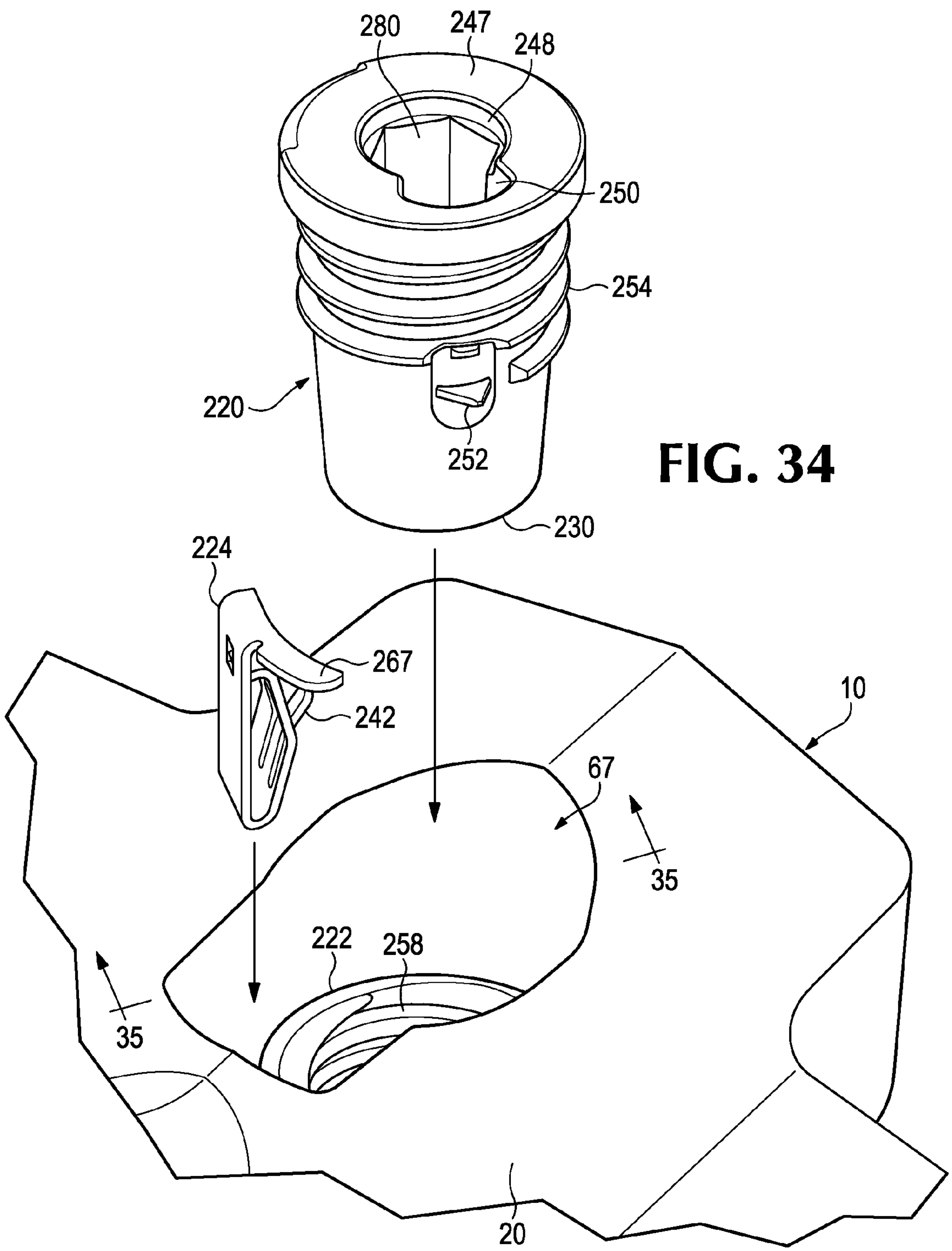
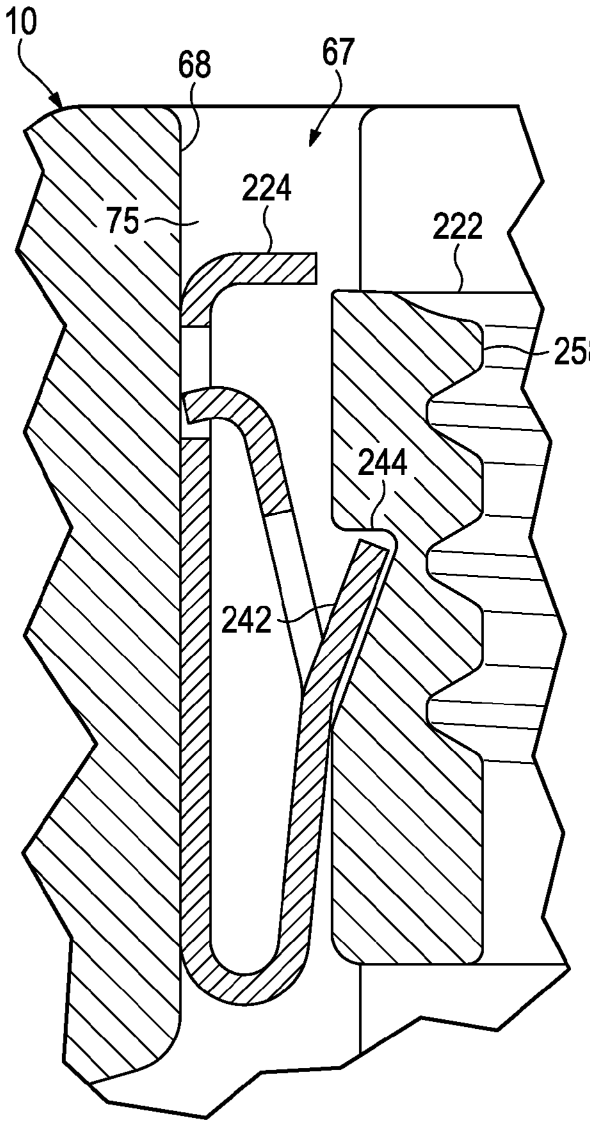


FIG. 34



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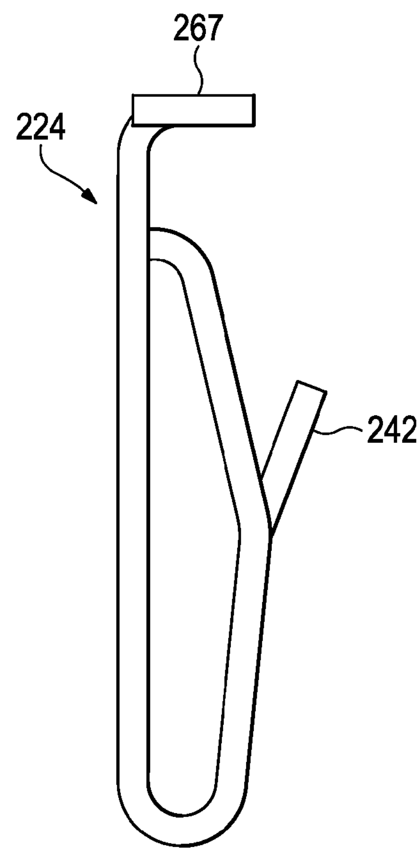

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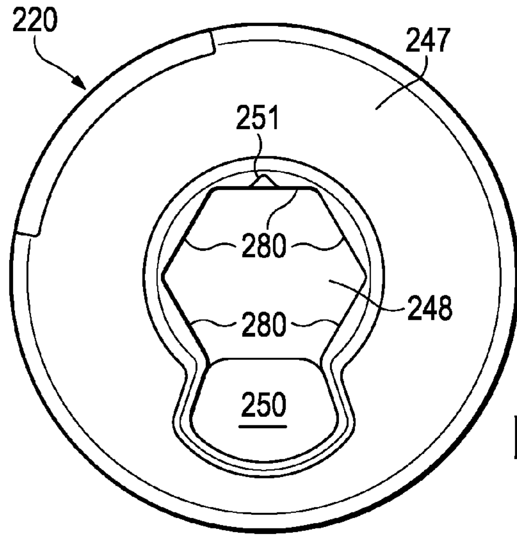

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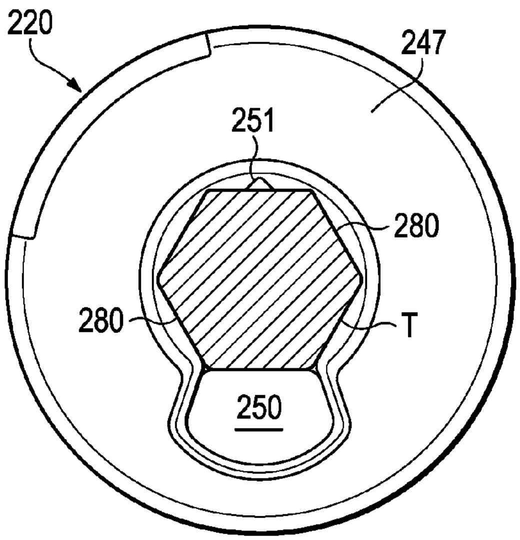

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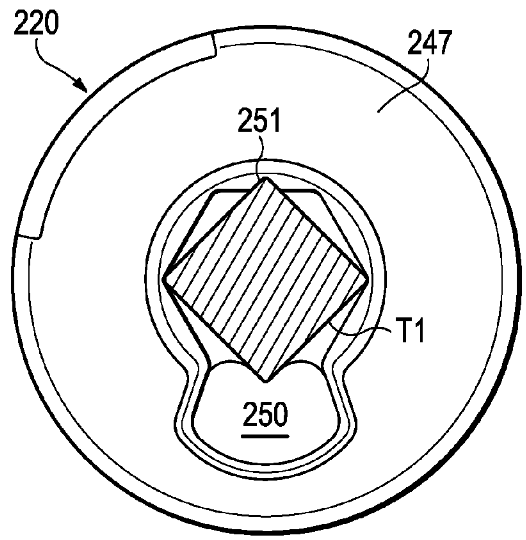
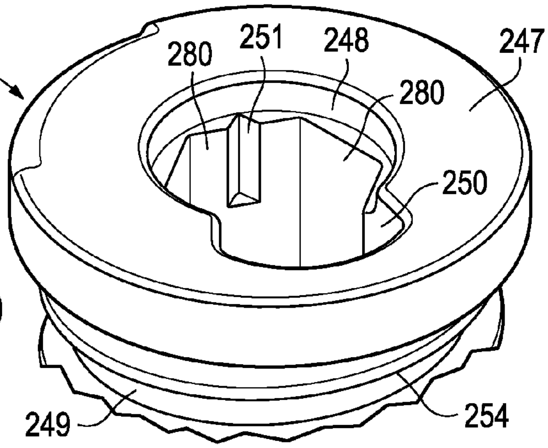


FIG. 39

FIG. 40



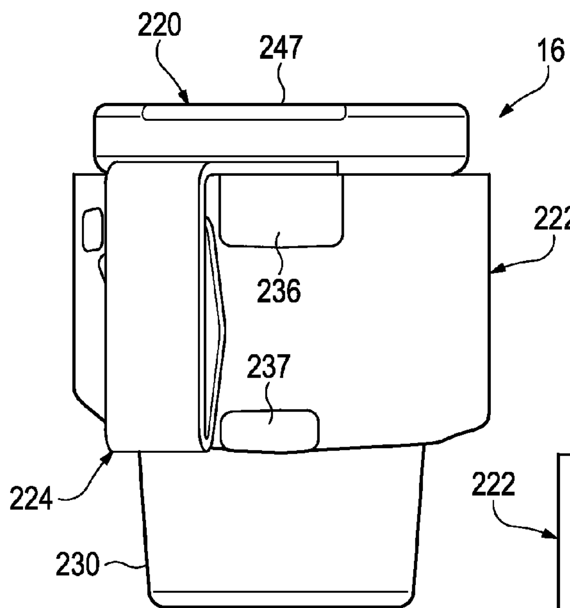


FIG. 41

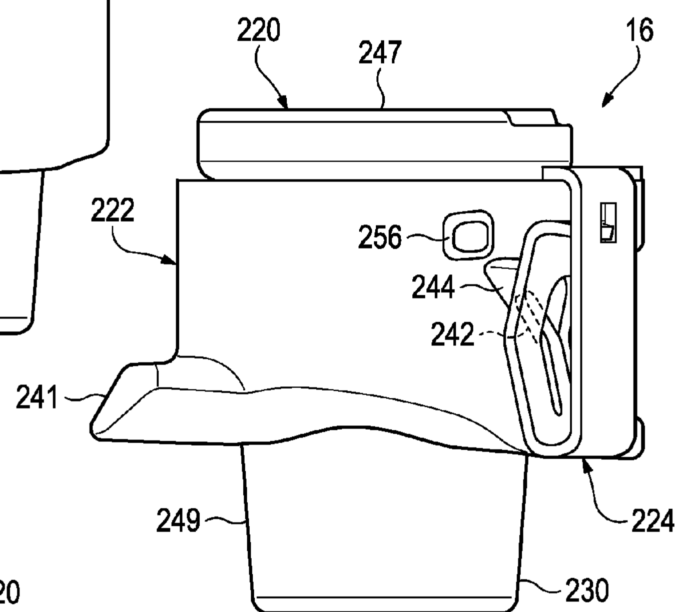


FIG. 42

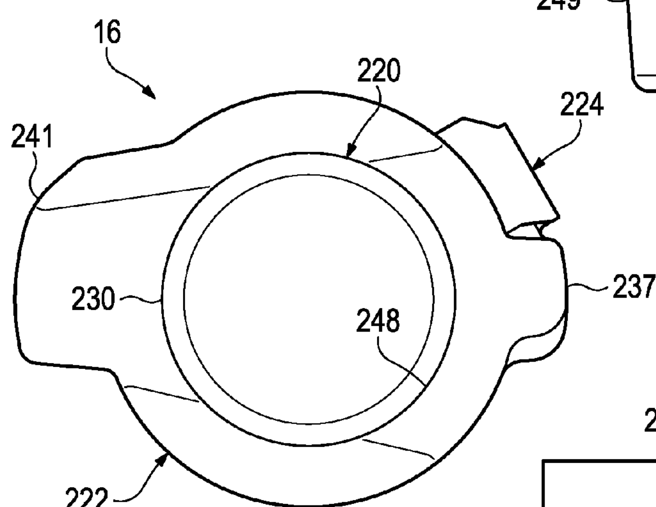


FIG. 43

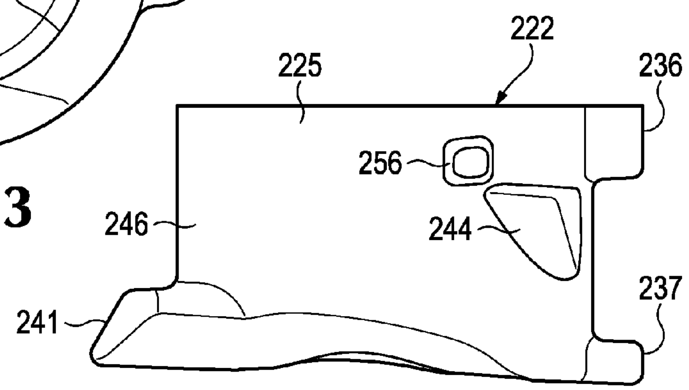


FIG. 44

WEAR ASSEMBLY

FIELD OF THE INVENTION

The present invention pertains to a wear assembly for use on various kinds of earth working equipment.

BACKGROUND OF THE INVENTION

In mining and construction, wear parts are commonly provided along the digging edge of excavating equipment such as buckets for dragline machines, cable shovels, face shovels, hydraulic excavators, and the like. The wear parts protect the underlying equipment from undue wear and, in some cases, also perform other functions such as breaking up the ground ahead of the digging edge. During use, the wear parts typically encounter heavy loading and highly abrasive conditions. As a result, they must be periodically replaced.

These wear parts usually comprise two or more components such as a base that is secured to the digging edge, and a wear member that mounts on the base to engage the ground. The wear member tends to wear out more quickly and is typically replaced a number of times before the base must also be replaced. One example of such a wear part is an excavating tooth that is attached to the lip of a bucket for an excavating machine. A tooth typically includes an adapter secured to the lip of a bucket and a point attached to the adapter to initiate contact with the ground. A pin or other kind of lock is used to secure the point to the adapter. Improvements in strength, stability, durability, safety, and ease of installation and replacement are desired in such wear assemblies.

SUMMARY OF THE INVENTION

The present invention pertains to a wear assembly for use on various kinds of earth working equipment including, for example, excavating machines and ground conveying means.

In one aspect of the invention, the wear assembly includes a base with a supporting portion, a wear member with a cavity into which the supporting portion is received, and a lock to releasably secure the wear member to the base. The supporting portion is formed with top and bottom recesses that receive complementary projections of the wear member. These recesses and projections include aligned holes so as to receive and position the lock centrally within the wear assembly and remote from the wear surface. This arrangement shields the lock from abrasive contact with the ground and lessens the risk of ejection or loss of the lock.

In another aspect of the present invention, the wear assembly includes a base with a supporting portion and a wear member with a cavity to receive the supporting portion. The fit between the supporting portion and the wear member includes stabilizing surfaces along each of the top, bottom and side walls in a unique configuration that creates a highly stable mounting of the wear member with improved penetrability.

In another aspect of the present invention, the wear member includes a wear indicator depression that opens in the nose-receiving cavity and is initially closed and spaced from the external wear surface, but which breaks through the wear surface when it is time to replace the wear member because of wear.

In another aspect of the invention, the wear member includes a hole for receiving the lock to secure the wear member to the base. The hole is defined by a wall that includes a retaining structure provided with an upper bearing surface and a lower bearing surface for contacting and retaining the

lock against upward and downward movement in the hole. In one preferred construction, a passage is provided in the hole to enable a lock or lock component to fit into the hole as an integral unit and be positioned to contact the upper and lower bearing surfaces of the retaining structure.

In another aspect of the invention, the lock includes a mounting component provided with a securing structure for attachment within a hole in the wear member. The securing structure cooperates with a retaining structure within the hole to resist movement of the mounting component in and out of the hole during use. The mounting component defines a threaded opening for receiving a threaded pin that is used to releasably hold the wear member to the base. The separate mounting component can be easily manufactured and secured within the wear member for less expense and higher quality than forming the threads directly in the wear member. The mounting component can be mechanically held within the hole in the wear member to resist axial movement in either direction so as to avoid unintended loss of the lock.

In another aspect of the invention, the lock includes a mounting component received and mechanically secured into a hole in the wear member to resist axial movement, a locking component movably received in the mounting component to releasably secure a wear member to a base, and a retainer to prevent release of the mounting component from the wear member.

In another aspect of the invention, the lock includes threaded components that are mechanically secured to a hardened steel wear member. The lock component can be adjusted between two positions with respect to the wear member: a first position where the wear member can be installed or removed from the base, and a second position where the wear member is secured to the base by the lock. The lock is preferably securable to the wear member by mechanical means at the time of manufacture so that it can be shipped, stored and installed as an integral unit with the wear member, i.e., with the lock in a "ready to install" position. Once the wear member is placed onto the base, the lock is moved to a second position to retain the wear member in place for use in an earth working operation.

In another aspect of the invention, a lock for releasably securing a wear member to earth working equipment includes a threaded pin with a socket in one end for receiving a tool to rotate the pin. The socket includes facets for receiving the tool, and a clearance space in lieu of one of the facets to better avoid and clean out earthen fines from the socket.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. **1** is a perspective view of a wear assembly in accordance with the present invention.

FIG. **2** is a side view of the wear assembly.

FIG. **3** is a perspective view of a base for the wear assembly.

FIG. **4** is a front view of the base.

FIG. **5** is a top view of the base.

FIG. **6** is a side view of the base.

FIG. **7** is a cross-sectional view taken along line **7-7** in FIG. **5**.

FIG. **8** is a top view of a wear member for the wear assembly.

FIG. **9** is a cross-sectional view taken along line **9-9** in FIG. **8**.

FIG. **10** is a cross-sectional view taken along line **10-10** in FIG. **8**.

FIG. **10**A is a cross-sectional view taken along line **10**A-**10**A in FIG. **8**.

FIG. **11** is a rear view of the wear member.

FIG. **12** is a cross-sectional view taken along line **12-12** in FIG. **11**.

FIG. **13** is a cross-sectional view taken along line **13-13** in FIG. **11**.

FIG. **14** is an exploded, perspective view of the wear assembly.

FIG. **15** is a partial side view of the base.

FIG. **16** is a cross-sectional view taken along line **16-16** in FIG. **15**.

FIG. **17** is a cross-sectional view taken along line **17-17** in FIG. **15**.

FIG. **18** is a cross-sectional view taken along line **18-18** in FIG. **15**.

FIG. **19** is a cross-sectional view taken along line **19-19** in FIG. **15**.

FIG. **20** is a cross-sectional view taken along line **20-20** in FIG. **15**.

FIG. **21** is a partial side view of the wear assembly.

FIG. **22** is a cross-sectional view taken along line **22-22** in FIG. **21**.

FIG. **23** is a cross-sectional view taken along line **23-23** in FIG. **21**.

FIG. **24** is a cross-sectional view taken along line **24-24** in FIG. **21**.

FIG. **25** is a cross-sectional view taken along line **25-25** in FIG. **21**.

FIG. **26** is a cross-sectional view taken along line **26-26** in FIG. **21**.

FIG. **27** is a perspective view of a lock of the wear assembly.

FIG. **28** is an exploded, perspective view of a lock of the wear assembly.

FIG. **29** is a cross-sectional view taken along line **29-29** in FIG. **2** with the lock in the release position.

FIG. **30** is a partial cross-sectional view taken along line **29-29** in FIG. **2** with the lock in the locked position.

FIG. **31** is a partial perspective view of the wear member.

FIG. **32** is a partial perspective view of the wear member with a mounting component of the lock partially installed.

FIG. **33** is a partial perspective view of the wear member with the mounting component installed in the wear member.

FIG. **34** is a partial perspective view of the wear member with an integral mounting component of the lock and a retainer and pin ready for installation.

FIG. **35** is a cross-sectional view taken along line **35-35** in FIG. **34**.

FIG. **36** is a side view of a retainer of the lock.

FIG. **37** is a top view of the pin.

FIGS. **38** and **39** are each a top view of the pin with tools shown in the socket.

FIG. **40** is a partial perspective view of the pin.

FIG. **41** is a front view of the lock.

FIG. **42** is a side view of the lock.

FIG. **43** is a bottom view of the lock.

FIG. **44** is a side view of the mounting component of the lock.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

The present invention pertains to a wear assembly for various kinds of earth working equipment including, for example, excavating equipment and ground conveying equipment. Excavating equipment is intended as a general term to refer to any of a variety of excavating machines used in mining, construction and other activities, and which, for example, include dragline machines, cable shovels, face shovels, hydraulic excavators, and dredge cutters. Excavating equipment also refers to the ground-engaging components of these machines such as the bucket or the cutter head. The digging edge is that portion of the equipment that leads the contact with the ground. One example of a digging edge is the lip of a bucket. Ground conveying equipment is also intended as a general term to refer to a variety of equipment that is used to convey earthen material and which, for example, includes chutes and mining truck beds. The present invention is suited for use along the digging edge of excavating equipment in the form of, for example, excavating teeth and shrouds. Additionally, certain aspects of the present invention are also suited for use along the expanse of a wear surface in the form of, for example, runners.

Relative terms such as front, rear, top, bottom and the like are used for convenience of discussion. The terms front or forward are generally used to indicate the normal direction of travel during use (e.g., while digging), and upper or top are generally used as a reference to the surface over which the material passes when, for example, it is gathered into the bucket. Nevertheless, it is recognized that in the operation of various earth working machines the wear assemblies may be oriented in various ways and move in all kinds of directions during use.

In one example, a wear assembly **14** in accordance with the present invention is an excavating tooth that attaches to a lip **15** of a bucket (FIGS. **1**, **2** and **14**). The illustrated tooth **14** includes an adapter **19** welded to lip **15**, an intermediate adapter **12** mounted on adapter **19**, and a point (also called a tip) **10** mounted on base **12**. While one tooth construction is shown, other tooth arrangements using some or all of the aspects of the invention are possible. For example, adapter **19** in this embodiment is welded to lip **15**, but it could be mechanically attached (e.g., by a Whisler-style lock assembly). In addition, the base could be an integral portion of the excavating equipment rather than a separately attached component. For example, adapter **19** could be replaced by an integral nose of a cast lip. Although in this application, for purposes of explanation, the intermediate adapter **12** is referred to as the base and the point **10** as the wear member, the intermediate adapter **12** could be considered the wear member and the adapter **19** the base.

Adapter **19** includes a pair of legs **21**, **23** that straddle lip **15**, and a forwardly projecting nose **18**. The intermediate adapter **12** includes a rearwardly-opening cavity **17** to receive nose **18** at the front end of adapter **19** (FIGS. **1**, **2**, **5** and **14**). Cavity **17** and nose **18** are preferably configured as disclosed in U.S. Pat. No. 7,882,649 which is incorporated herein by reference, but other nose and cavity constructions could be used. Adapter **12** includes a forwardly-projecting nose **48** to mount point **10**. Point **10** includes a rearwardly-opening cavity **26** to receive nose **48**, and a front end **24** to penetrate the ground. Lock **16** is used to secure wear member **10** to base **12**, and base **12** to nose **18** (FIGS. **1**, **2** and **14**). In this example, the locks to secure both the wear member **10** to base **12**, and the base **12** to nose **18** are the same. Nevertheless, they could be dimensioned differently, have different constructions, or could be completely different locks. With the use of an intermediate adapter, the tooth is well suited for use on larger machines, but could also be used on smaller machines. As an alternative, a point as the wear member could be secured directly onto adapter **19** as the base.

Wear member **10**, in this embodiment, has a generally wedge-shaped configuration with a top wall **20** and a bottom wall **22** that converge to a narrow front end **24** to engage and penetrate the ground during operation of the equipment (FIGS. **1**, **2** and **8-14**). A cavity **26** opens in the rear end **28** of wear member **10** for receiving base **12**. Cavity **26** preferably includes a front end portion **30** and a rear end portion **32**. The front or working portion **27** of wear member **10** is that portion forward of cavity **26**. The rear or mounting portion **29** of wear member **10** is that portion that includes cavity **26**.

The front end portion **30** of cavity **26** (FIGS. **10-13**) includes upper and lower stabilizing surfaces **34**, **36**. Stabilizing surfaces **34**, **36** axially extend substantially parallel to the longitudinal axis **42** of cavity **26** for improved stability under vertical loads (i.e., loads that include a vertical component). The term "substantially parallel" in this application means actually parallel or at a small diverging angle (i.e., about 7 degrees or less). Accordingly, stabilizing surfaces **34**, **36** axially extend at an angle of about 7 degrees or less to longitudinal axis **42**. Preferably, the stabilizing surfaces axially diverge rearwardly from the longitudinal axis at an angle of about five degrees or less, and most preferably at an angle of 2-3 degrees.

Stabilizing surfaces **34**, **36** oppose and bear against complementary stabilizing surfaces **44**, **46** on the nose **48** of base **12** (FIG. **24**). Stabilizing surfaces **44**, **46** are also substantially parallel to longitudinal axis **42** when the components are assembled together (FIGS. **3-7**, **14-16** and **24**). The bearing of stabilizing surfaces **34**, **36** in cavity **26** against stabilizing surfaces **44**, **46** on nose **48** provides a stable mounting of wear member **10** under vertical loads. Vertical loads applied to the front end **24** of wear member **10** urge the wear member (if not restricted by the nose and lock) to roll forward and off of the nose. Stabilizing surfaces (i.e., surfaces that are substantially parallel to the longitudinal axis **42**) resist this urge more effectively than surfaces with greater axial inclinations, and provide a more stable mounting of wear member **10** on nose **48**. A more stable mounting enables the use of a smaller lock and results in less internal wear between the parts.

Front end portion **30** of cavity **26** further includes side bearing surfaces **39**, **41** to contact complementary side bearing surfaces **45**, **47** on nose **48** to resist side loads (i.e., loads with a side component). Side bearing surfaces **39**, **41** in cavity **26** and side bearing surfaces **45**, **47** on nose **48** preferably axially extend substantially parallel to longitudinal axis **42** for greater stability in the mounting of wear member **10**. These front side bearing surfaces **39**, **41**, **45**, **47** cooperate with rear bearing surfaces that also resist side loads (as discussed below). In the preferred embodiment, the front bearing surfaces **34**, **36**, **39**, **41** in cavity **26** are each formed with slight lateral concave curvature for better resisting shifting loads and loads from all directions. Front bearing surfaces **44-47** on nose **48** would have a complementary convex configuration. The front bearing surfaces in cavity **26** and on nose **48** could, however, be flat or formed with a different curvature.

Nose **48** of base **12** includes a rear or main portion **50** rearward of stabilizing surfaces **44**, **46** of the front end **52** (FIGS. **3-7** and **14-20**); the nose **48** is considered that portion of adapter **12** that is received into cavity **26** of wear member **10**. The main portion **50** generally has a "dog bone" configuration in cross section (FIGS. **18-20**) with a narrower central section **54** and larger or thicker side sections **56**. Such a construction resembles an I-beam construction in function, and provides an attractive balance of strength with reduced mass and weight. In the preferred embodiment, side sections **56** are the mirror image of each other. The side sections **56** gradually increase in thickness from front to back for increased strength and reduced stress in the design. The use of a nose **48** having a narrow center section **54** and enlarged side sections **56** provides the dual benefit of (i) the nose **48** having sufficient strength to withstand the heavy loading that may be encountered during operation, and (ii) positioning the lock **16** at a central location in the wear assembly **14** to shield it from abrasive contact with the ground during use and to reduce the risk of lock ejection. The central section **54** preferably represents about the central two thirds or less of the overall thickness (i.e., height) of the nose **48** along the same lateral plane. In a most preferred embodiment, the thickness of central section **54** is about 60% or less of the largest or overall thickness of nose **48** along the same lateral plane.

Central section **54** is defined by a top surface **58** and a bottom surface **60**. Top and bottom surfaces **58**, **60** preferably axially extend substantially parallel to longitudinal axis **42**, but they could have a greater inclination. Top surface **58**, on each side, blends into an inner surface **62** on side sections **56**. Inner surfaces **62** are laterally inclined upward and outward from top surface **58** to partially define the upper part of side sections **56**. Likewise, inner surfaces **64** are laterally inclined downward and outward from bottom surface **60** to partially define the lower part of side sections **56**. Inner surfaces **62** are each laterally inclined to top surface **58** at an angle α of about 130-140 degrees to resist both vertical and side loading on wear member **10**, and reduce stress concentrations during loading (FIG. **20**). However, they could be at an angle outside of this range (e.g., about 105-165 degrees) if desired. Inner surfaces **64** are preferably mirror images of inner surfaces **62**, but they could be different if desired. The preferred ranges of inclinations are the same for both sets of inner surfaces **62**, **64**. The most preferred inclination for each inner surface **62**, **64** is at an angle α of 135 degrees. In some constructions, it may be preferred to have each inner surface **62**, **64** inclined at an angle α of more than 135 degrees to the adjacent top or bottom surface to provide greater resistance to vertical loads. Inner surfaces **62**, **64** are preferably stabilizing surfaces that each axially extend substantially parallel to the longitudinal axis **42** to better resist vertical loads and provide a stable mounting of the wear member **10** on base **12**.

A central hole **66** is formed in central section **54** that opens in top and bottom surfaces **58**, **60** (FIGS. **3**, **5**, **7**, **19**, **25** and **29**), though it could open only in top surface **58** if desired. The downward extension of hole **66** through bottom surface **60** reduces the build-up of earthen fines in the hole and enables an easier cleaning out of the fines in the hole. Top wall **20** of wear member **10** includes a through-hole **67** that aligns with hole **66** when wear member **10** is mounted on nose **48** (FIGS. **1**, **9**, **10**A, **13**, **14**, **25** and **29**). Lock **16** is received into the holes **66**, **67** to hold wear member **10** to base **12** (FIGS. **25**, **29** and **30**). The details of preferred lock **16** are provided below. However, other locks could be used to secure wear member **10** to base **12**. As examples, alternative locks could be in the form disclosed in U.S. Pat. No. 7,578,081 or U.S. Pat. No. 5,068,986, each of which are incorporated herein by reference. The shape of the aligned holes in the wear member and the base in instances of using alternative locks would, of course, be different than illustrated herein to accommodate the different locks.

Hole **67** in wear member **10** is defined by a wall **68** that preferably surrounds the lock **16** (FIG. **31**). Wall **68** includes a retaining structure **69** that extends laterally along part of the wall to define an upper bearing surface **71** and a lower bearing surface **73**. Bearing surfaces **71**, **73** are each contacted by lock **16** to hold the lock in the hole and resist inward and outward vertical forces applied to the lock during shipping, storage, installation and use of the wear member so as to better resist lock ejection or loss. In a preferred embodiment, retaining structure **69** is formed as a radial projection extending into hole **66** from wall **68** wherein the bearing surfaces **71**, **73** are formed as upper and lower shoulders. Alternatively, retaining structure **69** could be formed as a recess (not shown) in perimeter wall **68** with upper and lower bearing surfaces that face each other. A passage **75** is provided vertically along wall **68** in hole **67** to enable the insertion of lock **16** and the engagement of retaining structure **69**, i.e., with lock **16** in bearing contact with both the upper and lower bearing surfaces **71**, **73**. In the illustrated embodiment, no hole is formed in the bottom wall **22** of the wear member **10**; but a hole could be so formed to enable reversible mounting of point **10**. Also, if desired, base **12** could be reversibly mounted on nose **18** if the fit between the base **12** and nose **18** permit it. In the illustrated embodiment, base **12** cannot be reversibly mounted on nose **18**.

In a preferred embodiment, retaining structure **69** is essentially a continuation of wall **68** that is defined by a first relief **77** above or outside of the retaining structure **69**, a second relief **79** below or inside of the retaining structure **69**, and passage **75** at the distal end **81** of retaining structure **69**. Reliefs **77**, **79** and passage **75**, then, define a continuous recess **83** in perimeter wall **68** about retaining structure **69**. The end walls **87**, **89** of reliefs **77**, **79** define stops for the positioning of lock **16**. A recess **85** is preferably provided along an inside surface **91** of cavity **26** to function as a stop during the insertion of a mounting component of lock **16** as described below.

Cavity **26** in wear member **10** has a shape that complements nose **48** (FIGS. **9**, **10**, **10**A, **24-26** and **29**). Accordingly, the rear end **32** of the cavity includes an upper projection **74** and a lower projection **76** that are received into the upper and lower recesses **70**, **72** in nose **48**. Upper projection **74** includes an inside surface **78** that opposes top surface **58** on nose **48**, and side surfaces **80** that oppose and bear against inner surfaces **62** on nose **48**. Preferably there is a gap between inside surface **78** and top surface **58** to ensure contact between side surfaces **80** and inner surfaces **62**, but they could be in contact if desired. Side surfaces **80** are laterally inclined to match the lateral inclination of inner surfaces **62**. Side surfaces **80** axially extend substantially parallel to the longitudinal axis **42** to match the axial extension of inner surfaces **62**.

Lower projection **76** is preferably the mirror image of upper projection **74**, and includes an inside surface **82** to oppose bottom surface **60**, and side surfaces **84** to oppose and bear against inner surfaces **64**. In cavity **26**, then, inside surface **78** faces inside surface **82** with gap **86** in between the two inside surfaces **78**, **82** that is slightly larger than the thickness of central section **54** of nose **48**. The thickness (or height) of gap **86** is preferably within the middle two thirds of the overall thickness (or height) of the cavity (i.e., the largest height) **26** along the same lateral plane, and is most preferred within the middle 60% or less of the overall thickness of the cavity along the same lateral plane. Side surfaces **80**, **84** are laterally inclined away from the respective inside surfaces **78**, **82**, and axially extending substantially parallel to the longitudinal axis **42** to define upper and lower rear stabilizing surfaces for the point. The front stabilizing surfaces **34**, **36** cooperate with rear stabilizing surfaces **80**, **84** to stably support wear member **10** on nose **48**. For example, a downward vertical load L**1** on the front end **24** of wear member **10** (FIG. **2**) is primarily resisted by front stabilizing surface **34** in cavity **26** bearing against front stabilizing surface **44** on nose **48**, and rear stabilizing surfaces **84** in cavity **26** bearing against rear stabilizing surfaces **64** on nose **48** (FIGS. **24-26** and **29**). The axial extension of these stabilizing surfaces **34**, **44**, **64**, **86** (i.e., that they are axially substantially parallel to the longitudinal axis **42**) minimizes the forward, downward tendency to roll that load L**1** urges on wear member **10**. Likewise, an opposite upward load L**2** on front end **24** (FIG. **2**) would be primarily resisted by front stabilizing surface **36** in cavity **26** bearing against front stabilizing surface **46** on nose **48**, and rear stabilizing surfaces **80** in cavity **26** bearing against rear stabilizing surfaces **62** on nose **48** (FIGS. **24-26** and **29**). In the same way as noted above, stabilizing surfaces **36**, **46**, **62**, **84** stably support wear member **10** on base **12**.

The bearing contact between side surfaces **80** and inner surfaces **62**, and between side surfaces **84** and inner surfaces **64**, resists both vertical loads and loads with lateral components (called side loads). It is advantageous for the same surfaces to resist both vertical and side loads because loads are commonly applied to wear members in shifting directions as they are forced through the ground. With the laterally inclined stabilizing surfaces, bearing between the same surfaces can continue to occur even if a load shifts, for example, from more of a vertical load to more of a side load. With this arrangement, movement of the point on the nose is lessened, which leads to reduced wearing of the components.

A hollow portion **88**, **90** is provided to each side of each of the upper and lower projections **74**, **76** in cavity **26** for receiving side sections **56** of nose **48** (FIGS. **9**, **10**, **12**, **13**, **25**, **26** and **29**). The hollow portions **88**, **90** complement and receive side sections **56**. The upper hollow portions **88** are defined by side surfaces **80** on projection **74**, and outer surfaces **92**. The lower hollow portions **90** are defined by side surfaces **84** of projection **76**, and outer surfaces **94**. Outer surfaces **92**, **94** are generally curved and/or angular in shape to complement the top, bottom and outside surfaces of the side sections **56**.

In the preferred construction, each sidewall **100** of nose **48** is provided with a channel **102** (FIGS. **18-20**). Each channel is preferably defined by inclined channel walls **104**, **106** giving the channel a generally V-shaped configuration. Channels **102** each preferably has a bottom wall **107** to avoid a sharp interior corner, but they could be formed without a bottom wall (i.e., with a blend joining walls **104**, **106**) if desired. Channel walls **104**, **106** are each preferably inclined to resist both vertical and side loads. In a preferred construction, the channel walls **104**, **106** diverge to define an included angle β of about 80-100 degrees (preferably about 45 degrees to each side of a central horizontal plane), though the angle could be outside of this range. Channel walls **104**, **106** preferably each axially extend parallel to the longitudinal axis **42**.

The opposite sides **98** of cavity **26** define projections **108** that complement and are received into channels **102**. Projections **108** include bearing walls **110**, **112** that oppose and bear against channel walls **104**, **106** to resist vertical and side loading. Projections **108** preferably extend the length of sidewalls **98**, but they could be shorter and received in only portions of channels **102**. Bearing walls **110**, **112** preferably match the lateral inclination of channel walls **104**, **106**, and axially extend substantially parallel to longitudinal axis **42**.

While any opposing parts of the wear member **10** and base **12** may engage one another during use, the engagement of surfaces **34**, **36**, **44**, **46**, **62**, **64**, **80**, **84**, **104**, **106**, **110**, **112** are intended to the primary bearing surfaces to resist both vertical and side loading. The contact of front wall **114** of cavity **26** against front face **116** of nose **48** are intended to be the primary bearing surfaces resisting axial loads (i.e., loads with components that are parallel to longitudinal axis **42**).

Wear member **10** preferably includes laterally spaced recesses **123**, **125** in top wall **20** and corresponding laterally spaced recesses **127**, **129** in bottom wall **22** at the rear end **28** (FIGS. **1**, **2**, **10**, **14** and **26**). Nose **48** preferably includes cooperative recesses **130**, **132**, **134**, **136** (FIGS. **1-3**, **5**, **6** and **26**) that are laterally offset from recesses **123**, **125**, **127**, **129** on wear member **10** so that the rear end **28** of wear member **10** interlocks with the rear end **138** of nose **48** (FIGS. **1**, **2** and **26**). Side segments **124** of wear member **10** are received in side recesses **130**, **136** of base **12**, top segment **126** of wear member **10** is received in top recess **132** in base **12**, and bottom segment **128** of wear member **10** is received in bottom recess **134** of base **12** when the wear member is fully seated on nose **48**. Likewise, the lower and upper base segments **140**, **142** are received in cooperative recesses **123**, **125**, **127**, **129** of wear member **10**. This interlocked engagement of wear member **10** and base **12** resists loads during use. Nevertheless, other constructions could be used or the interlocking construction could be omitted, i.e., with rear end **28** having a continuous construction without recesses **123**, **125**, **127**, **129**.

Wear member **10** preferably includes a wear indicator depression **170** that opens in cavity **26** (FIG. **26**). In the illustrated example, wear indicator depression **170** is a slot formed in bottom wall **22** proximate rear end **28**, though other positions can be used. Depression **170** has a bottom surface **172** to define a depth that is spaced from wear surface **13** when wear member **10** is new. When depression **172** breaks through wear surface **13** during use, it provides a visual indicator to the operator that it is time to replace wear member.

Locks **16** are preferably used to secure wear member **10** to base **12**, and base **12** to nose **18** (FIGS. **1**, **2** and **14**). In the preferred construction, one lock **16** in top wall **20** is provided to hold wear member **10** to base **12**, and one lock **16** in each side wall **151** of base **12** is provided to hold base **12** to adapter **19**. Alternatively, two locks could be used to secure wear member **10** to base **12** and one lock to hold base **12** to adapter **19**. A hole **146** is provided on each side **151** of base **12** for receiving the respective lock **16**. Each hole **146**, then, has the same construction as described above for hole **67**. Further, a hole **161**, like hole **66**, is provided in the opposite sides **163** of nose **18**. Holes **161** are preferably closed, but could be interconnected through nose **18**. The locks though could have a wide variety of constructions. The lock securing base **12** to nose **18** could, for example, be constructed such as disclosed in U.S. Pat. No. 5,709,043.

Lock **16** includes a mounting component or collar **222** and a retaining component or pin **220** (FIGS. **27-44**). Collar **222** fits in hole **67** of wear member **10** and includes a bore or opening **223** with threads **258** for receiving pin **220** with matching threads **254**. A retainer **224**, preferably in the form of a retaining clip, is inserted in hole **67** with collar **222** to prevent disengagement of the collar **222** from wear member **10**. Preferably, retainer **224** is inserted during manufacture of wear member **10** so that lock **16** is integrally coupled with wear member **10** (i.e., to define a wear member that integrally includes a lock) for shipping, storage, installation and/or use of the wear member. Such a construction reduces inventory and storage needs, eliminates dropping the lock during installation (which can be particularly problematic at night), ensures the proper lock is always used, and eases the installation of the wear member. Nevertheless, if desired, retainer **224** could be removed at any time to effect removal of lock **16**.

Collar **222** has a cylindrical body **225** with lugs **236**, **237** that project outward to contact and bear against bearing surfaces or shoulders **71**, **73** of retaining structure **69** to hold lock **16** in place in wear member **10**. To install collar **222**, body **225** is inserted into hole **67** from within cavity **26** such that lugs **236**, **237** is slid along passage or slot **75**, and then rotated so that lugs **236**, **237** straddle retaining structure **69** (FIGS. **32** and **33**). Collar **222** is preferably translated into hole **67** until flange **241** is received in recess **85** and abuts against wall **93** of recess **85** (FIG. **32**). Collar **222** is then rotated until lugs **236**, **237** abut stops **87**, **89** (FIG. **33**). The rotation of collar **222** is preferably approximately 30 degrees so that lugs **236**, **237** move into upper reliefs **77**, **79** and abut stops **87**, **89**. Other stop arrangements are possible, e.g., the collar could have a formation abut end wall **81** or have only one lug engage the stop. In this position, lug **236** sets against upper bearing surface or shoulder **71**, and lug **237** against lower bearing surface or shoulder **73**. The engagement of lugs **236**, **237** against both sides of retaining structure **69** hold collar **222** in hole **67** even under load during digging. Further, the cooperation of outer lug **236** and flange **241** provide a resistive couple against cantilever loads applied to pin **220** during use.

Once collar **222** is in place, a retainer or clip **224** is inserted into passage **75** from outside wear member **10** (FIG. **34**). Preferably, retainer **224** is snap-fit into slot **75**, thereby preventing rotation of collar **222** so that lugs **236**, **237** are retained in reliefs **77**, **79** and against shoulders **71**, **73**. Retainer **224** is preferably formed of sheet steel with a bent tab **242** that snaps into a receiving notch **244** on an outer surface **246** of collar **222** to retain retainer **224** in wear member **10** (FIGS. **35** and **36**). The retainer allows collar **222** to be locked in wear member **10** for secure storage, shipping, installation and/or use, and thereby define an integral part of wear member **10**. Furthermore, retainer **224** preferably exerts a spring force against collar **222** to bias collar **222** to tighten the fit of collar **222** in hole **67**. A flange **267** is preferably provided to abut lug **236** and prevent over-insertion of the retainer.

The engagement of lugs **236**, **237** against shoulders **71**, **73** mechanically hold collar **222** in hole **67** and effectively prevent inward and outward movement during shipping, storage, installation and/or use of wear member **10**. A mechanical attachment is preferred because the hard, low alloy steel commonly used to manufacture wear members for earth working equipment generally lacks sufficient weldability. Collar **222** is preferably a single unit (one piece or assembled as a unit), and preferably a one piece construction for strength and simplicity. Retainer **224** is preferably formed of sheet steel as it does not resist the heavy loads applied during used. Retainer **224** is used only to prevent undesired rotation of collar **222** in hole **67** so as to prevent release of lock **16** from wear member **10**.

Pin **220** includes a head **247** and a shank **249** (FIGS. **28-30**, **34** and **37-40**). Shank **249** is formed with threads **254** along a portion of its length from head **247**. Pin end **230** is preferably unthreaded for receipt into hole **66** in nose **48**. Pin **220** is installed into collar **222** from outside wear member so that pin end **230** is the leading end and pin threads **254** engage collar threads **258**. A hex socket (or other tool-engaging formation) **248** is formed in head **247**, at the trailing end, for receipt of a tool T to turn pin **220** in collar **222**.

Preferably, hex socket **248** is provided with a clearance opening **250** in place of one facet (i.e., only five facets **280** are provided), to define a cleanout region (FIGS. **27**, **28**, **34** and **37-40**). Cleanout region **250** makes the resulting opening larger, and therefore less likely to retain impacted fines and grit that often packs such pockets and openings on ground-engaging portions of earth working equipment. Cleanout region **250** also provides alternate locations to insert tools to break up and pry out compacted fines. For example, a sharp chisel, pick, or power tool implement may be shoved, pounded, or driven into cleanout region **250** to begin breaking up compacted fines. Should any damage occur to the interior surfaces of cleanout region **250** during the process, the damage generally has no impact on the five active tool faces of hex engagement hole **48**. Once some of the compacted fines are broken out of cleanout region **250**, any compacted fines inside hex engagement hole **248** may be attacked from the side or at an angle, as accessed through cleanout region **250**.

An additional benefit of a lobe-shaped cleanout region is that the combination of a hex socket with a lobe-shaped cleanout region on one facet of the hex socket also creates a multiple-tool interface for pin **20**. For example, a hex socket sized for use with a ⅞-inch hex drive T (FIG. **38**), when elongated on one face, will allow a ¾-inch square drive T**1** to fit (FIG. **39**) as well. Optimal fit for such a square drive is obtained by forming a groove **251** in one facet of hex socket **248**, opposite cleanout region **250**. Other tools may fit as well, such as pry bars, if needed in the field when a hex tool is not available.

In one preferred embodiment, threaded pin **220** includes a biased latching tooth or detent **252**, biased to protrude beyond the surrounding thread **254** (FIGS. **29**, **30** and **34**). A corresponding outer pocket or recess **256** is formed in the thread **258** of collar **222** to receive detent **252**, so that threaded pin **220** latches into a specific position relative to collar **222** when latching detent **252** aligns and inserts with outer pocket **256**. The engagement of latching detent **252** in outer pocket **256** holds threaded pin **220** in a release position relative to collar **22**, which holds pin **220** outside of cavity **26** (or at least outside of hole **66** with sufficient clearance on nose **48**), so that the wear member **10** can be installed on (and removed from) nose **48**. The pin is preferably shipped and stored in the release position so that wear member **10** is ready to install. Preferably, latching detent **252** is located at the start of the thread on threaded pin **220**, near the pin end **230**. Outer pocket **256** is located approximately ½ rotation from the start of the thread on collar **222**. As a result, pin **220** will latch into shipping position after approximately ½ turn of pin **220** within collar **222**.

Further application of torque to pin **220** will squeeze latching detent **252** out of outer pocket **256**. An inner pocket or recess **260** is formed at the inner end of the thread of collar **222**. Preferably, the thread **258** of collar **222** ends slightly before inner pocket **260**. This results in an increase of resistance to turning pin **220** as pin **220** is threaded into collar **222**, when latching detent **252** is forced out of thread **258**. This is followed by a sudden decrease of resistance to turning pin **220**, as latching detent **252** aligns with and pops into the inner pocket. In use, there is a noticeable click or "thunk" as pin **220** reaches an end of travel within collar **222**. The combination of the increase in resistance, the decrease in resistance, and the "thunk" provides haptic feedback to a user that helps a user determine that pin **220** is fully latched in the proper service position. This haptic feedback results in more reliable installations of wear parts using the present combined collar and pin assembly, because an operator is trained to easily identify the haptic feedback as verification that pin **220** is in the desired position to retain wear member **10** on base **12**. The use of a detent **252** enables pin **220** to stop at the desired position with each installation unlike traditional threaded locking arrangements.

Preferably, latching detent **252** may be formed of sheet steel, held in place within a sump **262** within pin **220**, resiliently fixed in place inside an elastomer **264**. Sump **262** extends to open into cleanout region **250**. The elastomer contained in sump **262** also may extend into cleanout region **250**, when latching detent **252** is compressed during rotation of pin **220**. Conversely, the elastomer contained in sump **262** forms a compressible floor for cleanout region **250**, which may aid in the breakup and removal of compacted fines from cleanout region **250**. Elastomer **264** may be molded around latching detent **252** so that elastomer **264** hardens in place and bonds to latching detent **252**. The resulting subassembly of detent **252** and elastomer **264** may be pressed into place through cleanout region **250**, and into sump **262**. A preferred construction of latching detent **252** includes a body **266**, a protrusion **268**, and guide rails **270**. Protrusion **268** bears against a wall of sump **262**, which keeps latching detent **252** in proper location relative to thread **254**. Guide rails **270** further support latching detent **252**, while allowing compression of latching detent **252** into sump **262**, as discussed above.

When pin **220** is installed into collar **222**, it is rotated ½ turn to the release position for shipping, storage and/or installation of wear member **10**. The wear member containing integrated lock **16** is installed onto nose **48** of base **12** (FIG. **29**). Pin **220** is then preferably rotated 2½ turns until pin end **230** is fully received into hole **66** in the locked or service position (FIG. **30**). More or fewer rotations of threaded pin **220** may be needed, depending on the pitch of the threads, and on whether more than one start is provided for the threads. The use of a particularly coarse thread requiring only three full rotations of threaded pin **220** for full locking of a wear member **10** to base **12** has been found to be easy to use in field conditions, and reliable for use under the extreme conditions of excavation. Furthermore, the use of a coarse helical thread is better in installations where the lock assembly will become surrounded by compacted fines during use.

Lock **16** is located within the upper recess **70** between side sections **56** for protection against contact with the ground and wear during use (FIGS. **25** and **30**). The positioning of lock **16** deep in wear assembly **14** helps shield the lock from wear caused by the ground passing over wear member **10**. Preferably, lock **16** is recessed with hole **67** so that it remains shielded from moving earthen material over the life of the wear member. In a preferred example, pin **220** in the locked position is in the bottom 70% or lower in hole **67**. Earthen material will tend to accumulate in hole **67** above lock **10** and protect the lock from undue wear even as wear member **10** wears. Further, the lock is generally centrally located in wear assembly with pin end **230** located at or proximate the center of hole **66** in the locked position. Positioning the lock closer to the center of nose **18** will tend to reduce ejection loads applied to the lock during use of the wear member, and especially with vertical loads that tend to rock the wear member on the base.

Pin **20** may be released using a ratchet tool or other tool to unscrew pin **220** from collar **222**. While pin **220** can be removed from collar **222**, it need only be backed up to the release position. Wear member **10** can then be removed from nose **48**. The torque of unscrewing pin **220** may exert substantial torsion loads on collar **222**, which loads are resisted by stops **77** and **79**, providing a strong and reliable stop for lugs **236** and **237**.

The mounting component **222** of lock **16** defines a threaded bore **223** for receiving a threaded securing pin **220** that is used to releasably hold wear member **10** to base **12** (and base **12** to adapter **19**). The separate mounting component **222** can be easily machined or otherwise formed with threads, and secured within the wear member for less expense and higher quality threads as compared to forming the threads directly in the wear member. The steel used for wear member **10** are very hard and it is difficult to cast or otherwise form screw threads into hole **67** for the intended locking operation. The relatively large size of wear member **10** also makes it more difficult to cast or otherwise form screw threads in hole **67**. The mounting component **222** can be mechanically held within the hole in the wear member to resist axial movement in either direction (i.e., that is in and out of hole **67**) during use so as to better resist unintended loss of the lock during shipping, storage, installation and use. On account of the hard steel typically used for wear member **10**, mounting component **222** could not be easily welded into hole **67**.

The use of a lock in accordance with the present invention provides many benefits: (i) a lock integrated into a wear member so that the lock ships and stores in a ready to install position for less inventory and easier installation; (ii) a lock that requires only common drive tools such as a hex tool or ratchet driver for operation, and requires no hammer; (iii) a lock with easy tool access; (iv) a lock with clear visual and haptic confirmation of correct installation; (v) a new lock provided with each wear part; (vi) a lock that is positioned for easy access; (vii) a lock with a simple intuitive universally understood operation; (vii) a permanent mechanical connection between components of differing geometric complexity creates a finished product with features and benefits extracted from specific manufacturing processes; (viii) a lock integration system built around simple castable feature where the integration supports high loads, requires no special tools or adhesives and creates a permanent assembly; (ix) a lock with a hex engagement hole elongated on one facet allowing easier cleanout of soil fines with simple tools; (x) a lock located with a central part of the wear assembly to protect the lock from wear and reduce the risk of lock ejection; (xi) a lock with reaction lugs on the lock collar to carry system loads perpendicular to bearing faces; (xii) a retaining clip installed at the manufacturing source that holds the collar into the wear member while also biasing the collar against the load bearing interface and taking slack out of the system; (xiii) a design approach that simplifies casting complexity while supporting expanded product functionality; (xiv) a design approach whereby critical fit surfaces in the lock area need only be ground to fit one part which could act as a gage; and (xv) a design that fits within standard plant processes.

Lock **16** is a coupling arrangement for securing two separable components in an excavating operation. The system consists of a pin **220** received in a hole **66** in a base **12** and a collar **222** mechanically retained in the wear member **10**. The collar contains features supportive of integrated shipment, load transmission, lock installation and lock removal. The collar is secured to the wear member with a retainer **224** which acts upon two lugs **236**, **237** at the perimeter of the collar maintaining the lugs in an optimal load bearing orientation. The retainer also tightens the fit between components. The pin **220** helically advances through the center of the collar **222** between two low energy positions created by an elastomer backed latching mechanism. The first position keeps ½ turn of thread engaged between the collar and the pin for retention during shipment. The pin **220** advances into the second low energy position after rotating 2½ turns ending in a hard stop signaling that the system is locked. When the wear member **10** requires changing, the pin **220** is rotated counterclockwise and removed from the assembly allowing the wear member to slide free from the base.

While the illustrated embodiment is an excavating tooth, the features associated with the locking of wear member **10** on base **12** can be used in a wide variety of wear assemblies for earth working equipment. For example, runners can be formed with a hole, like hole **67**, and mechanically secured to a base defined on the side of a large bucket, a chute surface, a bed of a truck body and the like.

The disclosure set forth herein encompasses multiple distinct inventions with independent utility. While each of these inventions has been disclosed in its preferred form, the specific embodiments thereof as disclosed and illustrated herein are not to be considered in a limiting sense as numerous variations are possible. Each example defines an embodiment disclosed in the foregoing disclosure, but any one example does not necessarily encompass all features or combinations that may be eventually claimed. Where the description recites "a" or "a first" element or the equivalent thereof, such description includes one or more such elements, neither requiring nor excluding two or more such elements. Further, ordinal indicators, such as first, second or third, for identified elements are used to distinguish between the elements, and do not indicate a required or limited number of such elements, and do not indicate a particular position or order of such elements unless otherwise specifically stated.

The invention claimed is:

1. A wear member for attachment to earth working equipment to protect the equipment from wear during use, the wear member comprising a front end to contact the ground during operation of the earth working equipment, a rearwardly-opening cavity with a longitudinal axis to receive a base on the earth working equipment, the cavity including a central section along the longitudinal axis and a side section to each side of the central section, each said side section including an outer side and an inner side, the inner sides each connecting with the central section, each outer side having an inwardly-projecting lateral projection defined by an upper outer bearing surface and a lower outer bearing surface, the upper and lower outer bearing surfaces being laterally inclined toward each other in an inward direction and axially extending substantially parallel to the longitudinal axis, each inner side having an inside bearing surface above and below the central section, each inside bearing surface being laterally inclined inward and away from the outer side and axially extending substantially parallel to the longitudinal axis, the outer bearing surfaces and the inside bearing surfaces each bearing against complementary bearing surfaces on the base to resist vertical and side loads applied to the wear member during use, the central section including an upper surface and a lower surface, the upper surface extending between and connecting the upper inside bearing surfaces, the lower surface extending between and connecting the lower inside bearing surfaces, the upper and lower surfaces being spaced apart to define a gap therebetween, the gap having a height between the upper and lower surfaces that is less than two-thirds of the overall height of the cavity, and at least one of the upper and lower surfaces including a hole for receiving a lock to secure the wear member to the earth working equipment.

2. A wear member in accordance with claim **1** wherein the cavity includes a front end portion including a front wall facing rearward, an upper stabilizing surface and a lower stabilizing surface, the upper and lower stabilizing surfaces face toward each other and axially extend rearward substantially parallel to the longitudinal axis from the front wall, and the upper and lower stabilizing surfaces bear against complementary surfaces on the base during use.

3. A wear member in accordance with claim **1** which includes an external wear surface to contact the ground during use, and a depression that opens in the cavity and extends outward partially through the wear member toward the wear surface as a wear indicator that is exposed in the wear surface when the wear member needs replacing.

4. A wear assembly for attachment to earth working equipment to protect the equipment from wear during use, the wear assembly comprising: a base secured to the earth working equipment, the base including a hole; a wear member including an external wear surface to contact the ground during operation of the earth working equipment, a rearwardly-opening cavity with a longitudinal axis to receive the base on the earth working equipment, the cavity including a central section along the longitudinal axis and a side section to each side of the central section, each said side section including an outer side and an inner side, the inner sides each connecting with the central section, each outer side having an inwardly-projecting lateral projection defined by an upper outer bearing surface and a lower outer bearing surface, the upper and lower outer bearing surfaces being laterally inclined toward each other in an inward direction and axially extending substantially parallel to the longitudinal axis, each inner side having an inside bearing surface above and below the central section, each inside bearing surface being laterally inclined inward and away from the outer side and axially extending substantially parallel to the longitudinal axis, the outer bearing surfaces and the inside bearing surfaces each bearing against complementary bearing surfaces on the base to resist vertical and side loads applied to the wear member during use, the central section including an upper surface and a lower surface, the upper surface extending between and connecting the upper inside bearing surfaces, the lower surface extending between and connecting the lower inside bearing surfaces, the upper and lower surfaces being spaced apart to define a gap therebetween, the gap having a height between the upper and lower surfaces that is less than two-thirds of the overall height of the cavity, and at least one of the upper and lower surfaces including a hole that aligns with the hole in the base; and a lock received in the holes in the wear member and the base to releasably secure the wear member to the earth working equipment.

5. A wear assembly in accordance with claim **4** wherein the lock includes a leading end and a trailing end, the base includes a nose received into the cavity of the wear member, the nose having a top side and a bottom side, and, when the lock is inserted into the holes in the wear member and the base, the leading end is in the hole in the base is at about the midpoint of the base between the top side and the bottom side and the trailing end is remote from the wear surface.

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